

# Variable Displacement Open Loop Circuit Axial Piston Pumps

**PV Series** 







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#### **TECHNICAL INFORMATION**





Quick Reference Data Chart

nominal pressure 350 bar max. pressure 420 bar

ଘଙ୍ଗାଲ 420 bar

- 1 New type of swash plate and large servo piston with strong bias spring achieves fast response, reduce the noise due to active decompression of system at down stroke.
- 2 Nine pistons and new precompression technology (precompression filter volume) result in unbeaten low outlet flow pulsation.
- 3 Complete compensator program offers multiple controls.
- 4 Rigid and FEM-optimized body design for lowest noise level.
- 5 Thru drive for 100% nominal torque.
- 6 Pump combinations (multiple pumps) of same size and model and mounting interface for basically all metric or SAE mounting interfaces.
- 7 Wide application in automobile industrial, ships, forging machines, tire machines, injection molding machines, machine tools, special-purpose machine.

		Pressure		Displacement		Pump Delivery ( 7 bar ) 100 PSI			APPRO	X. Noise	Levels	Speed		Weight	
Size	Model	nominal pressure	max. pressure	cm <sup>3</sup> /rev	ln³/rev		O RPM U.S. GPM		RPM U.S. GPM	Db(A) Full 70 bar (1 KSI)	Flow and 207 bar (3 KSI)	343 bar (5 KSI)	Max. RPM	Min. RPM	KG (LB)
	PV016			16	0.98	24	6.3	28.8	7.6						
1	PV020			20	1.2	30	7.9	36	9.5	56	60	68	2750		19 (42)
	PV023			23	1.4	34.5	9.1	41.4	10.9						, ,
	PV032	350	420	32	1.9	48	12.7	57.6	15.2						
	PV040			40	2.4	60	15.9	72	19	59		69			
2	PV046			46	2.8	69	18.2	82.2	21.9		62		2400		30 (66)
	PV056	280	350	56	3.41	84	22.1	100.8	26.6						
	PV065	250	315	65	3.96	97.5	25.7	117	30.9						
	PV063		420	63	3.8	94.5	25	113.4	30	66			2100	300	
	PV071	350 420		71	4.3	107	28.3	128.7	34				2100	-	60
3	PV080	330	420	80	4.8	120	31.7	144	38		70	74	2000		
3	PV092			92	5.6	138	36.5	165.6	43.8		70		1900		(132)
	PV110	250	280	110	6.7	165	43.6	198	52.3				1900		
	PV125	250	280	125	7.6	187.5	49.5	225	59.4				1900		
	PV140	250	420	140	8.5	210	55.5	252.1	66.6	70	74	76	2200		
4	PV180	350 4	420	180	11	270	71.3	324	85.6	71	75	77	2200		90 (198)
	PV210	300	350	210	12.8	315	83.1	378	99.8	73	77	79	2100		
5	PV270	350	420	270	16.5	405	107	486	128.4	77	79	89	1800		172 (379)

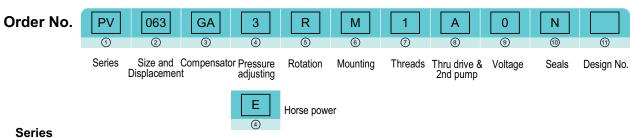
- 1 Outlet port is on the top, the pipe pressure should be less than 2 bar.
- 2 The usage of max. Pressure for each cycle never exceed 6 seconds.
  Please see the General Installation Information for hydraulic oil cleanliness manual.
- 3 We offers tandem pump or other types of pump connection. The mounting has Metric and SAE dimensions.



## **Variable Displacement** Open Loop Circuit Axial Piston Pump

## **PV Series**

#### **ORDER CODE**



Axial piston pump variable nominal pressure max. pressure displacement high pressure version 350 bar 420 bar	
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#### **Size and Displacement**

	Code		016	020	023	032	040	046	056	065	063	071	080	092	110	125	140	180	210	270
2	Size			1				2					3	3				4		5
)	Disast	cm <sup>3</sup> /rev	16	20	23	32	40	46	56	65	63	71	80	92	110	125	140	180	210	270
	Displ.	In <sup>3</sup> /rev	0.98	1.2	1.4	1.9	2.4	2.8	3.41	3.96	3.8	7.3	4.8	5.6	6.6	7.6	8.5	11	12.6	16.5

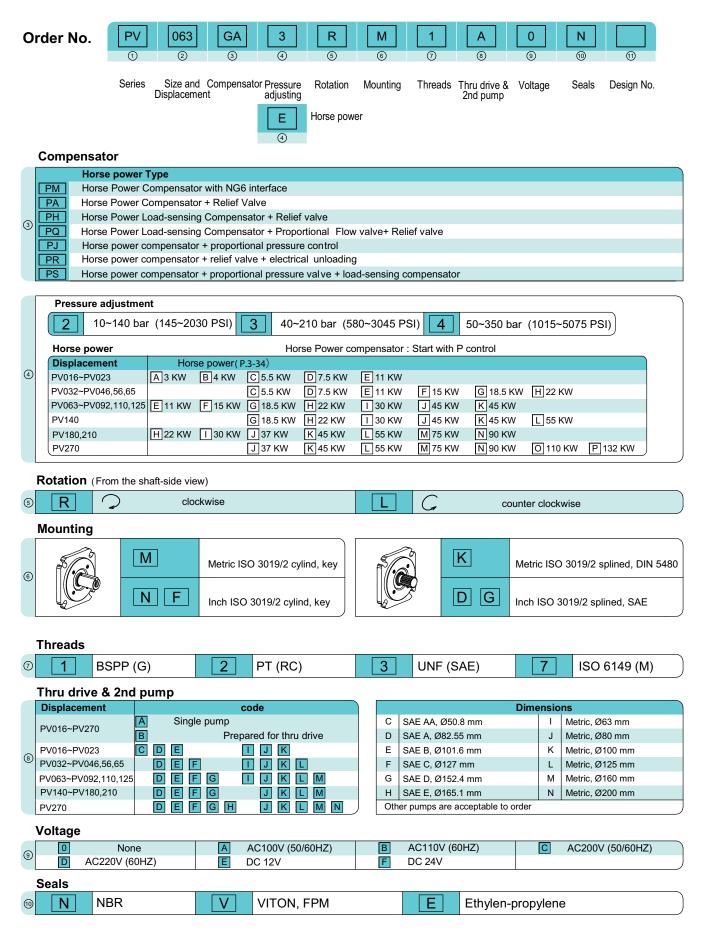
#### Compensator

	<u> </u>	ensator
	A0	Standard Pressure compensator
	LN	None Pressure Compensator (fixed displacement) (pressure protection required)
	LS	Electrical 2-stages Pressure control (pressure protection required)
	LC	Fixed Displacement 2 stages Flow compensator (pressure protection required)
		Remote Type
	GM	Remote Pressure compensator with NG6 interface
	GA	Remote Pressure compensator +Relief valve
	GJ	Remote Pressure compensator + Proportional Pressure valve
		Electrical unloading Type
	GR	Remote Pressure compensator + Electrical unloading
	GB	Remote Pressure compensator + 2-stages pressure control
	GC	Remote Pressure compensator + electrical unloading + 2-stages Pressure control
		Load-sensing Type
	HM	Load-sensing compensator with NG6 interface
	HA	Load-sensing compensator + Relief valve
	HJ	Load-sensing compensator + Proportional Pressure valve
		Load-sensing + Electrical unloading Type
3	HR	Load-sensing compensator + Electrical unloading
	HB	Load-sensing compensator + 2-stages Pressure control
	HC	Load-sensing compensator + Electrical unloading+ 2-stages Pressure control
		Proportional Pressure, flow Type
	HQ	Load-sensing compensator + Proportional Flow valve+ Relief valve
	LHK.	Load-sensing compensator + Proportional Pressure valve+ Proportional Flow valve
	BQ	None-stage Flow compensator (Cylinder)
		2-Valve Load-sensing Type
	VM	2-Valve load-sensing compensator with NG6 interface
	_VA_	2-Valve load-sensing compensator + Relief Valve
ļ		2-Valve load-sensing compensator + Proportional Pressure valve
		2-Valve Proportional Pressure, flow Type
	VQ	2-Valve load-sensing compensator + Proportional Flow valve+ Relief Valve
ı	VK	2-Valve load-sensing compensator + Proportional Pres sure valve+ Proportional Flow valve
		Proportional compensation Type
	FV	2-Valve load-sensing compensator + High Reacted Proportional Flow Valve+ Flow Feed Back+ Relief Valve
	FG	2-Valve load-sensing compensator + High Reacted Proportional Flow Valve+ Proportional Pressure+ Flow& Pressure Feed Back



## Open Loop Circuit Axial Piston Pump PV Series **Variable Displacement**

#### ORDER CODE (continued)





# Variable Displacement Open Loop Circuit Axial Piston Pump

## **PV Series**

### **ORDER CODE** (continued)

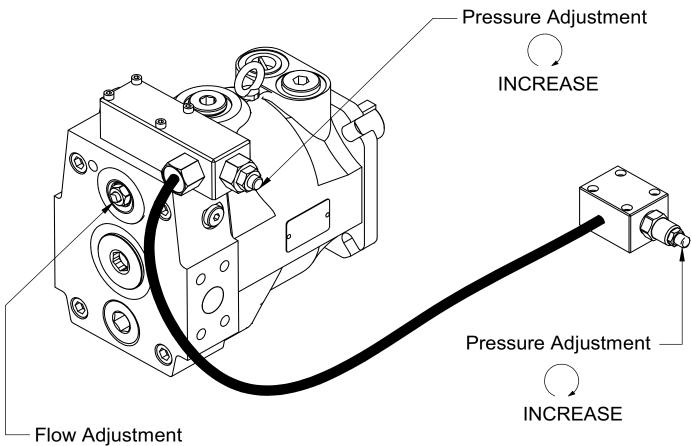
#### Compensator

A0	Standard Pressure compensator	
LN	None Pressure Compensator (fixed displacement) (pressure protection required)	
LS	Electrical 2-stages Pressure control (pressure protection required)	
LC	Fixed Displacement 2 stages Flow compensator (pressure protection required)	
	Remote Type	
GM	Remote Pressure compensator with NG6 interface	
GA	Remote Pressure compensator +Relief valve	
GJ	Remote Pressure compensator + Proportional Pressure valve	
	Electrical unloading Type	
GR	Remote Pressure compensator + Electrical unloading	
GB	Remote Pressure compensator + 2-stages pressure control	
GC	Remote Pressure compensator + electrical unloading + 2-stages Pressure control	
	Load consing Tune	
НМ	Load-sensing Type  Load-sensing compensator with NG6 interface	
HA	Load-sensing compensator + Relief valve	
HJ	Load-sensing compensator + Proportional Pressure valve	
110	Load sorising compensator - 1 reportional 1 ressure valve	
	Load-sensing + Electrical unloading Type	
HR	Load-sensing compensator + Electrical unloading	
НВ	Load-sensing compensator + 2-stages Pressure control	
HC	Load-sensing compensator + Electrical unloading+ 2- stages Pressure control	
	Proportional Pressure, Flow Type	
HQ	Load-sensing compensator + Proportional Flow valve+ Relief valve	
HK	Load-sensing compensator + Proportional Pressure valve+ Proportional Flow valve	
BQ	None-stage Flow compensator (Cylinder)	
	2-Valve Load-sensing Type	
VM	2-Valve load-sensing compensator with NG6 interface	
VA	2-Valve load-sensing compensator + Relief Valve	
VJ	2-Valve load-sensing compensator + Proportional Pressure valve	
	2 Value Dranational Dranaura Flaur Time	
VQ	2-Valve Proportional Pressure, Flow Type	
VK	2-Valve load-sensing compensator + Proportional Flow valve+ Relief Valve 2-Valve load-sensing compensator + Proportional Pressure valve+ Proportional Flow valve	
VIX	2 valve load sensing compensator - responding ressare valve - responding riow valve	
	Proportional Compensator Type	
FV	2-Valve load-sensing compensator + High Reacted Proportional Flow Valve+ Flow Feed Back+ Relief Valve	
FG	2-Valve load-sensing compensator + High Reacted Proportional Flow Valve+ Proportional Pressure+ Flow& Pressure Feed Back	
	2 Tail out Sandy Somponder - High reduced he portional flow valve in topolitonal flooding in lower resource feed back	
	Horse power Type	
PM	Horse Power Compensator with NG6 interface	
PA	Horse Power Compensator + Relief Valve	
PH	Horse Power Load-sensing Compensator + Relief valve	
PQ	Horse Power Load-sensing Compensator + Proportional Flow valve+ Relief valve	
PJ	Horse power compensator + proportional pressure control  Horse power compensator + relief valve + electrical unloading	
PR	Horse power compensator + relief valve + electrical unloading  Horse power compensator + proportional pressure valve + load-sensing compensator	Page
	Totas power compensator - proportional pressure valve - load-sensing compensator	. ugo



#### **COMPENSATOR**

### **Pressure and Flow Adjustement**



Release the screw nut before adjusting

**DECREASE** 



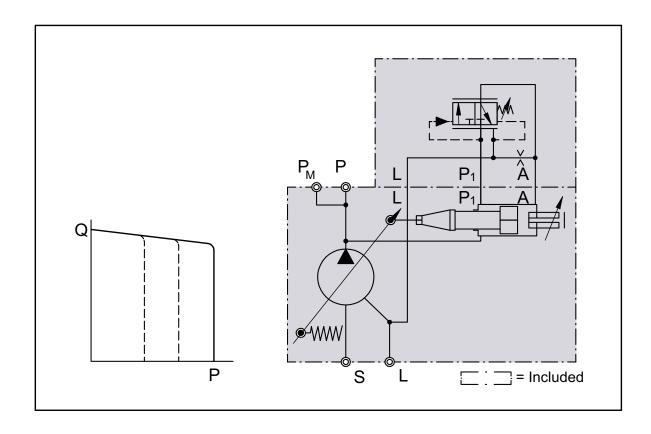
#### **COMPENSATOR**

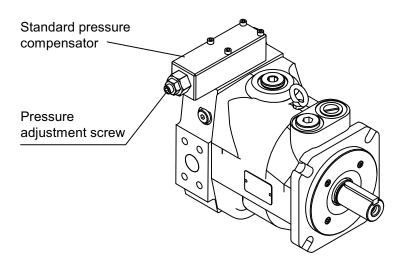
## **A0** Standard Pressure Compensator

The **A0** standard pressure compensator adjusts the pump displacement according to the actual need of the system in order to keep the pressure constant.

As long as the system pressure at outlet port P is lower than the set pressure (set as spring preload of the compensator spring) the working port A of the compensator valve is connected to the case drain and the piston area is unloaded.

Bias spring and system pressure on the annulus area keep the pump at full displacement. When the system pressure reaches the set pressure the compensator valve spol connects port P1 to A and builds up a pressure at the servo piston resulting in a downstroking of the pump. The displacement of the pump is controlled in order to match the flow requirement of the system.





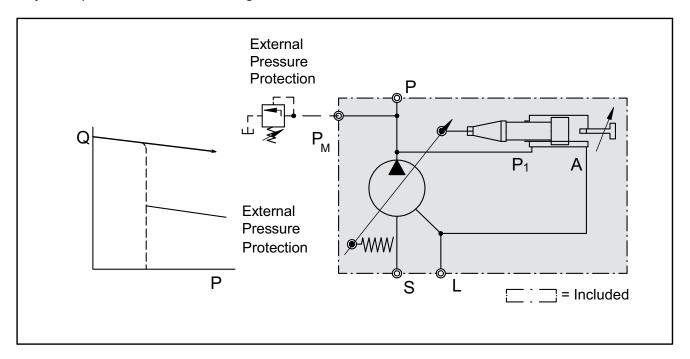


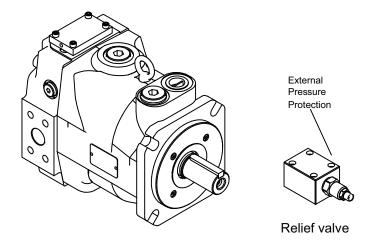
## LN None Pressure Compensator (fixed displacement - pressure protection required)

By using the system under stable displacement and pressure situation, standard pressure compensator can be omitted which helps cost down.

#### Notice:

External Pressure Protection is necessarily added at port PM to limit the pressure; otherwise the system pressure will be over high.





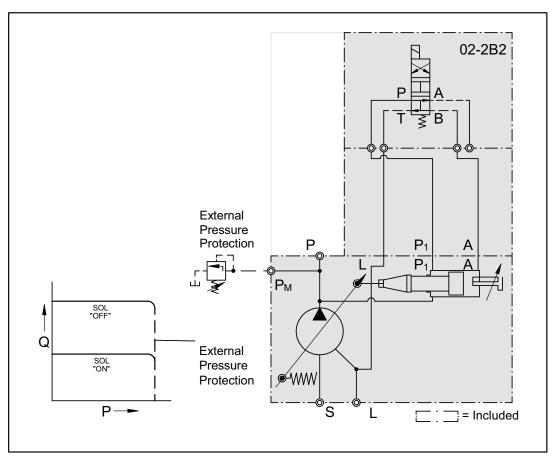


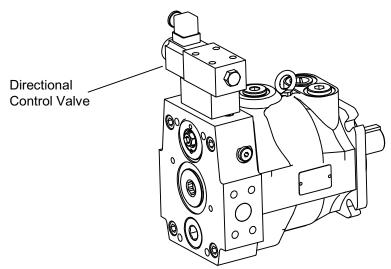
## LS Electrical Dual Flow Compensator (pressure protection required)

Control the hydraulic circuit change by switch Directional Control Valve. **LS** control is applied to two-stage stroke and different speed system.

#### Notice:

External Pressure Protection is necessarily added at port PM to limit the pressure; otherwise the system pressure will be over high.







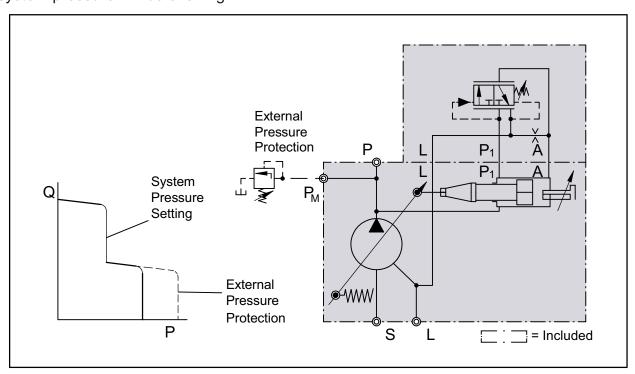
## LC Fixed Displacement 2 stages Flow Compensator (pressure protection required)

Control the hydraulic circuit change by using the system pressure setting to achieve the switch of big and small flow.

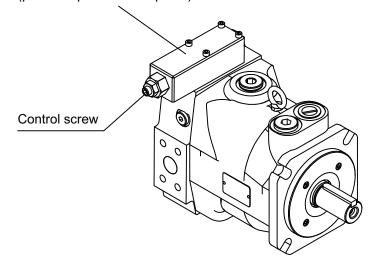
**LC** control is applied to two-stage stroke and different speed system.

#### Notice:

External Pressure Protection is necessarily added at port PM to limit the pressure; otherwise the system pressure will be over high.



Fixed Displacement 2 stages Flow compensator (pressure protection required)





## **GM** Remote Pressure Compensator with NG6 interface

Version **GM** of the remote pressure compensator provides on its top side an interface NG6, DIN24340 (CETOP 03 at RP35H, NFPA D03).

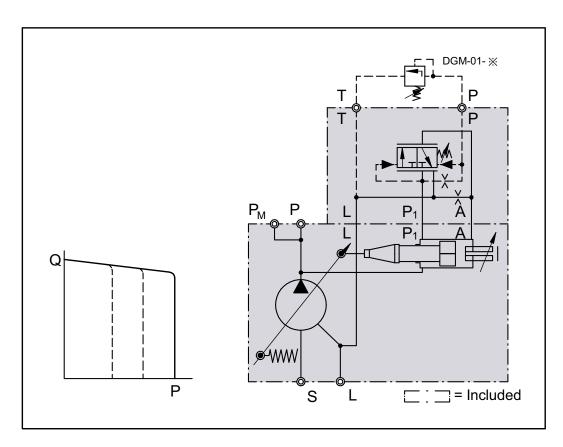
This interface allows a direct mounting of a pilot valve.

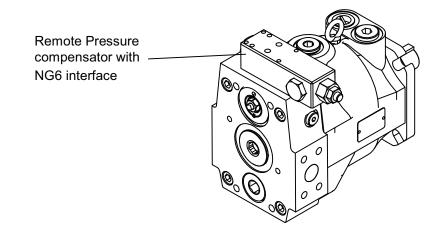
Beside manual or electrohydraulic operated valves, it is also possible to mount complete multiple pressure circuits directly on the compensator body.

We offers a variety of these compensator accessories ready to install.

All remote pressure compensator have a factory setting of 15 bar differential pressure.

With this setting, the controlled pressure at the pump outlet is higher than the pressure controlled by the pilot valve.







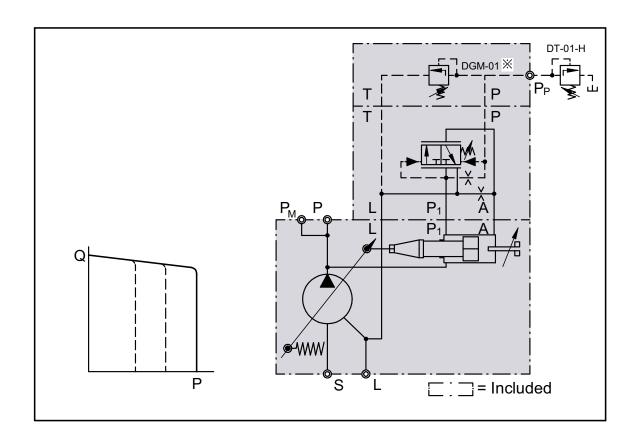
## **GA** Remote Pressure Compensator + Relief Valve

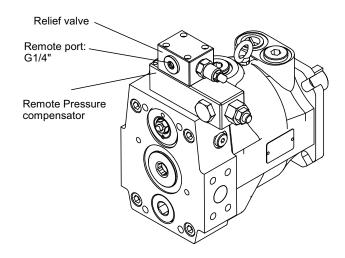
The pressure is set directly at the compensator spring, and the setting of remote pressure compensator can be achieved by any suitable pilot pressure valve connected to pilot port Pp.

The pilot valve can be installed remote from the pump in some distance.

That allows pressure setting, e.g. from the control panel of the machine.

The pilot flow supply is internal through the valve spool, and the pilot flow is 1~1.5 L/min.

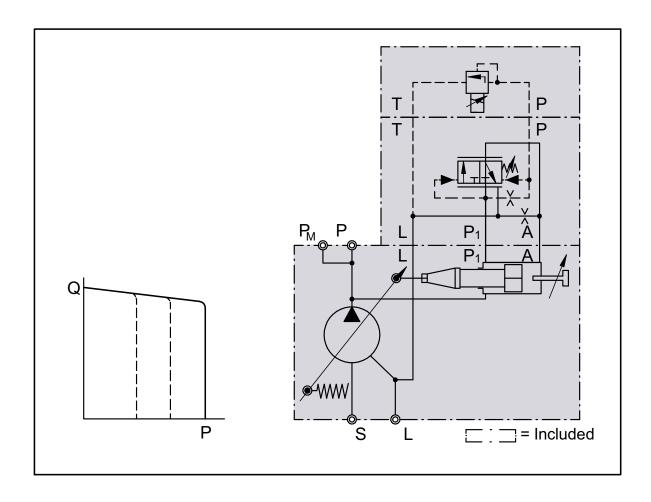


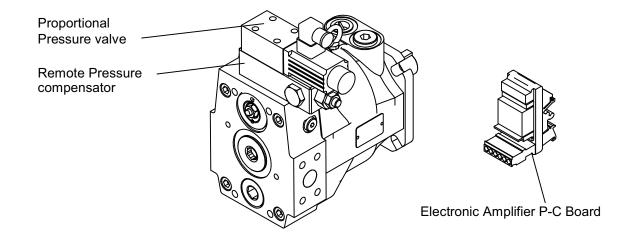




## **GJ** Remote Pressure Compensator + Proportional Pressure Valve

Fulfill the actual displacement and maintain the preset system pressure. By adding proportional pressure valve, Electrical proportional pressure control is available.



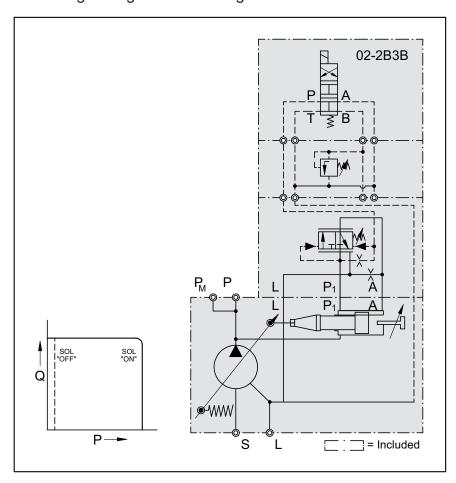


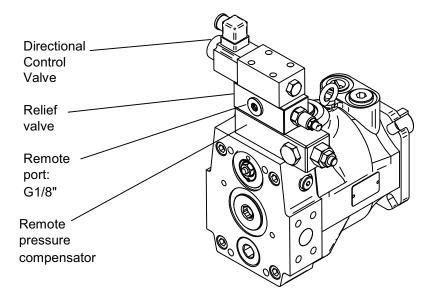


## **GR** Remote Pressure Compensator + Electrical Unloading

By adding a Relief Valve and a Directional Control Valve on the compensator makes the pump have both function.

**GR** control is for long unloading situation. When the system stops, oil temperature and noise maintain low level while being through the unloading.



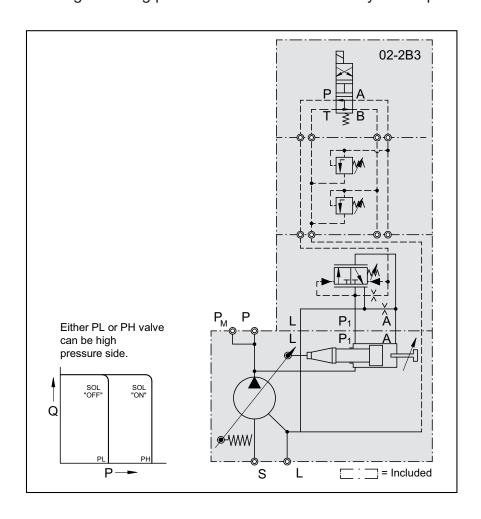


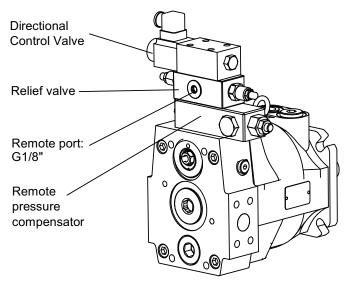


## **GB** Remote Pressure Compensator + 2-stage Pressure Control

By adding a Relief Valve and Directional Control Valve on the compensator makes it adjust two different stage limited pressure.

GB control is for two-stage working pressure under the constant cylinder speed.



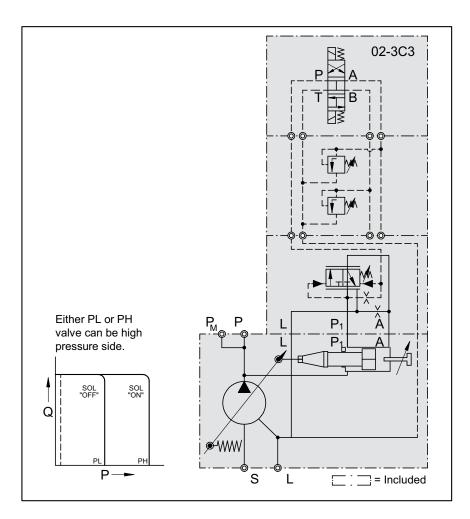


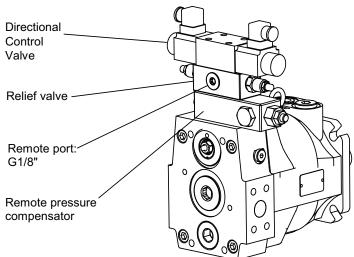


## **GC** Remote Pressure Compensator + Electrical Unloading + 2-stage Pressure Control

Control two different-stage limited pressure by adding Directional Control Valve, and unloading function.

When the system stops, oil temperature and noise maintain low level by unloading function. Usable for stable cylinder speed, two-stage pressure, and long unloading situation.







## **HM** Load-sensing Compensator with NG6 interface

Version **HM** of remote pressure compensator provides an interface NG6 on its top side.

The load-sensing compensator has an external pilot pressure supply.

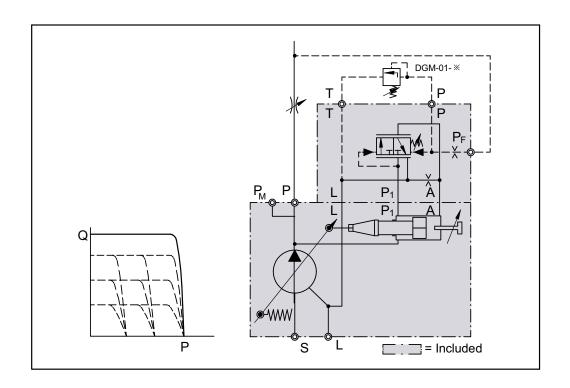
Factory setting for the differential pressure is 10 bar.

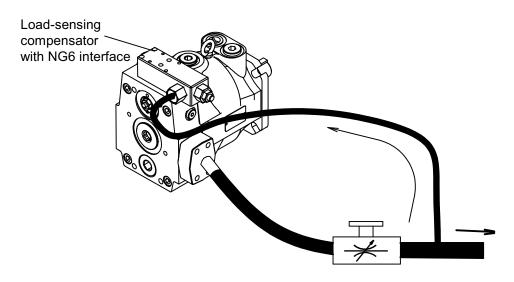
The input signal to the compensator is the differential pressure at the main stream resistor.

A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant.

A variable input speed or a varying load (-pressure) has consequently no influence on the output flow of the pump and the speed of the actuator.

By adding proportional pressure valve, Electrical proportional pressure control is available.





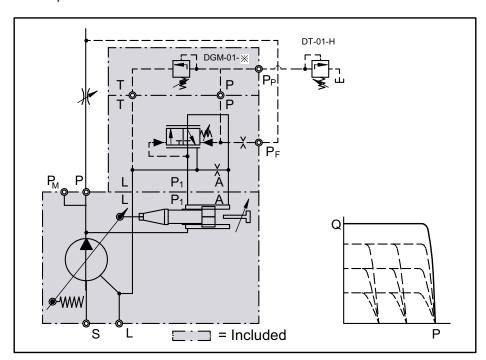


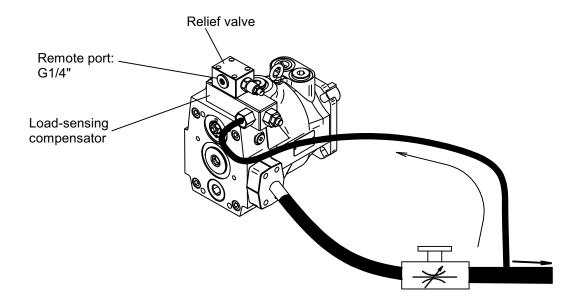
## **HA** Load-sensing Compensator + Relief Valve

The load-sensing compensator has an external pilot pressure supply.

Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant. A variable input speed or a varying load (-pressure) has consequent no influence on the output flow of the pump and the speed of the actuator. Relief valve has adjustment function.

The pilot valve can be installed remote form the pump in some distance. That allows pressure setting, e.g. form the control panel of the machine. The pilot flow supply is internal through the valve spool, and the pilot flow is 1-1.5 L/min.







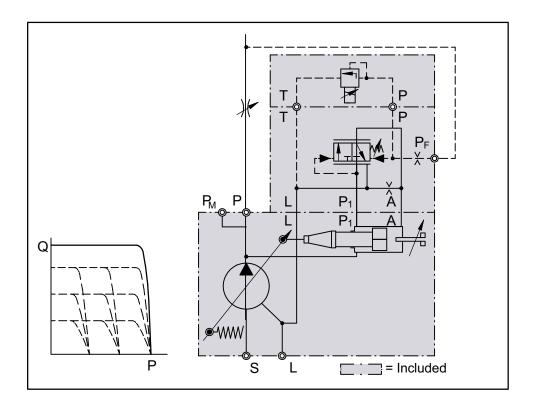
## **HJ** Load-sensing Compensator + Proportional Pressure Valve

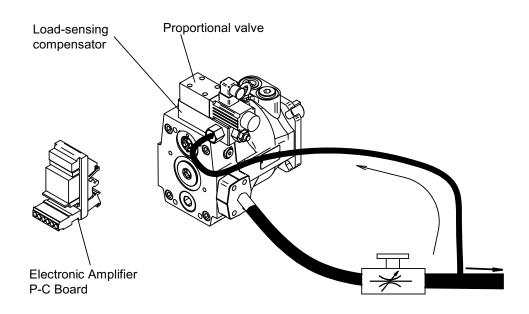
The load-sensing compensator has an external pilot pressure supply.

Factory setting for the differential pressure is 10 bar.

The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow and the speed of the actuator.

Proportional pressure valve is for Electrical proportional pressure control.







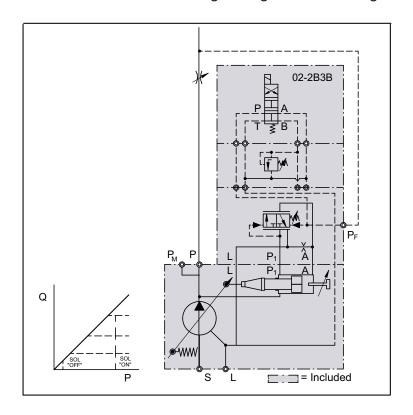
## HR Load-sensing Compensator + Electric Unloading

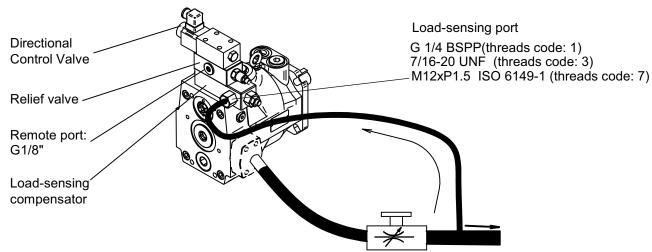
The load-sensing compensator has all external pilot pressure supply.

Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant. A variable input speed or a varying (load -pressure) has consequently on the output flow of the pump and speed of the actuator.

By adding a pilot orifice ( $\Phi$  0.8mm) and a pressure pilot valve pressure compensation can be added to the flow control function. See the circuit diagram below.

By adding a relief valve and a directional control valve on the compensator makes the pump have both function. **HR** control is for long unloading situation. When the system stops, oil temperature and noise maintain low level while being through the unloading.







## **HB** Load-sensing Compensator + 2-stage Pressure Control

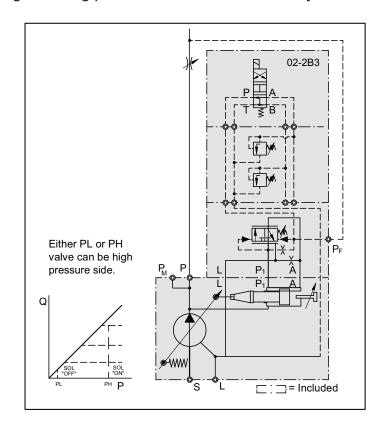
The load-sensing compensator has an external pilot pressure supply.

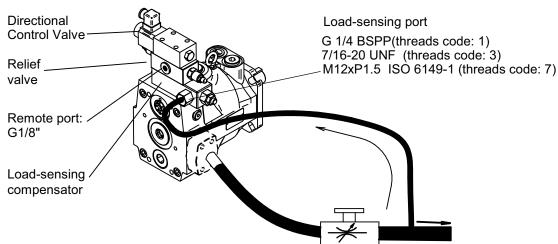
Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant. A variable input speed or a varying (load -pressure) has consequently on the output flow of the pump and speed of the actuator.

By adding a pilot orifice ( $\Phi$ 0.8mm) and a pressure pilot valve pressure compensation can be added to the flow control function. See the circuit diagram below.

By adding a relief valve and directional control valve on the compensator makes it adjust two different stage limited pressure.

**HB** control is for two-stage working pressure under the constant cylinder.







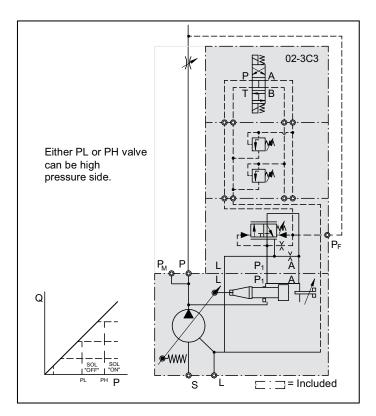
## **HC** Load-sensing Compensator + Electrical Unloading + 2-stage Pressure Control

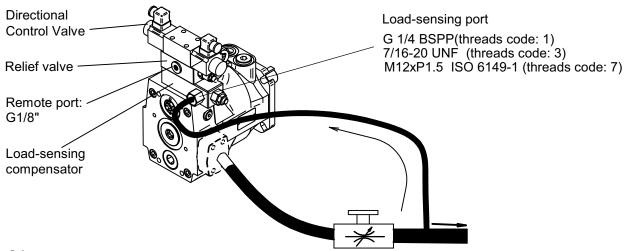
The load-sensing compensator has an external pilot pressure supply.

Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant. A variable input speed or a varying (load -pressure) has consequently on the output flow of the pump and speed of the actuator.

By adding a pilot orifice ( $\Phi$ 0.8mm) and a pressure pilot valve pressure compensation can be added to the flow control function. See the circuit diagram below.

By adding a relief valve and a directional control valve on the compensator makes the pump have both function. **HC** control is for long unloading situation. When the system stops, oil temperature and noise maintain low level while being through the unloading.



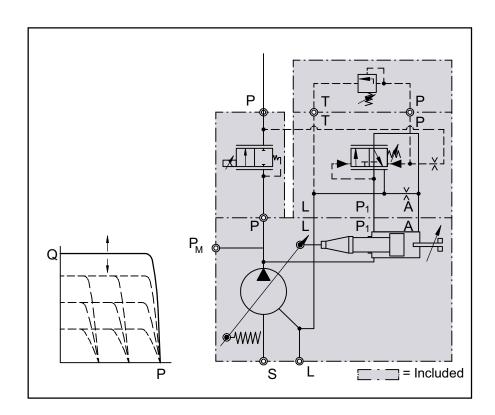


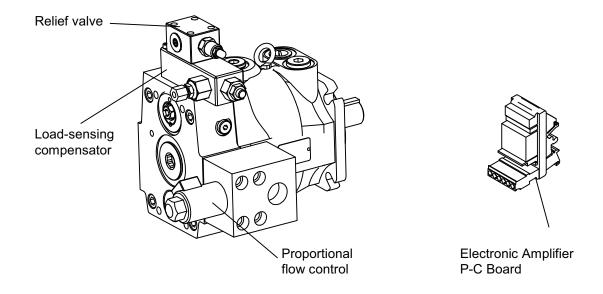


## **HQ** Load-sensing Compensator + Proportional Flow Valve + Relief Valve

The load-sensing compensator has an external pilot pressure supply. Factory setting for the differential pressure is 10 bar.

The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant. By adding proportional flow valve, Electrical proportional flow control is available.



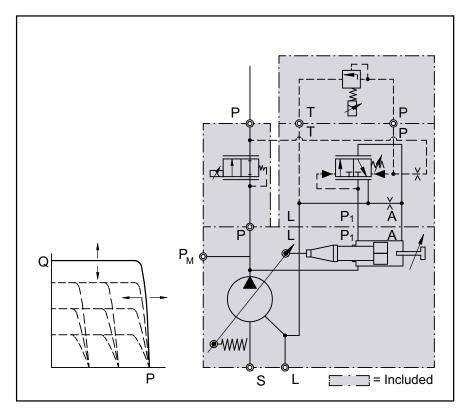


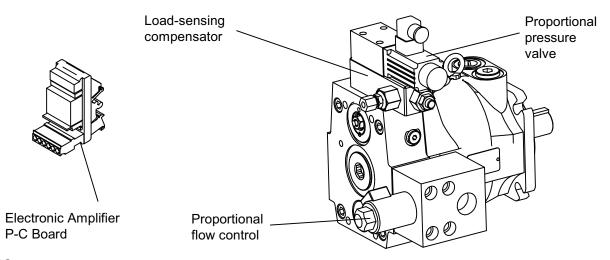


## **HK** Load-sensing Compensator + Proportional Pressure Valve +Proportional Flow Valve

**HK** is for saving energy. It offers the smallest pressure and flow according to the different requirement. The displacement is nearly zero when the system stands by, and the motor output is also nearly zero. When the system reaches setting pressure, the pump displacement will reduce by itself. It only needs to add the system required flow, and the pressure remains the same which control the oil temperature. Compared with vane pump, gear pump +PQ valve can save 30%-50% energy. The load-sensing compensator+ Proportional flow valve has all external pilot pressure supply. Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor.

A load-sensing compensator represents mainly a flow control for the pump output flow of the pump and the speed of the actuator. Proportional pressure valve is for electrical proportional pressure control.







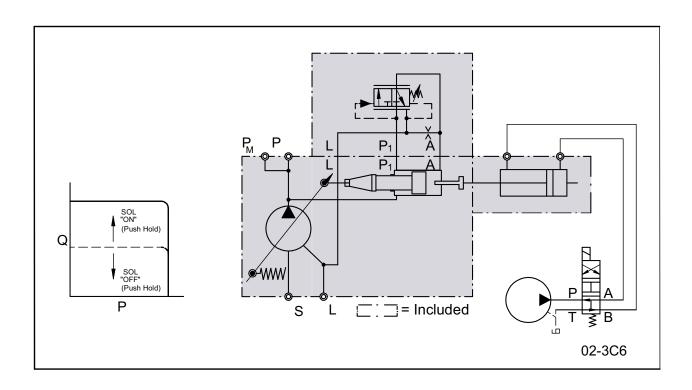
## **BQ** None-stage Flow Compensator (Cylinder)

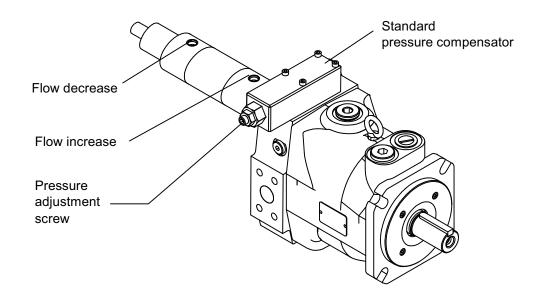
Using added cylinder, external pilot pressure controls the forward and backward, and push the swash plate to change directions and make the single-stage pressure control.

Displacement can be zero to Max, and pressure remains on setting pressure.

Automatous flow control is much easier to repair, and cheaper than electrical proportional flow control.

External control is more automated by using solenoid valve or hydraulic directional valve.



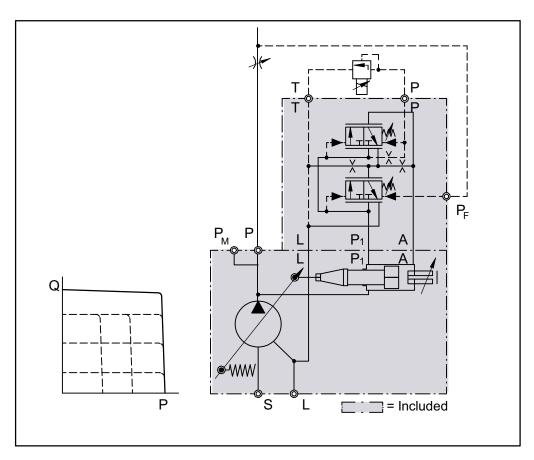


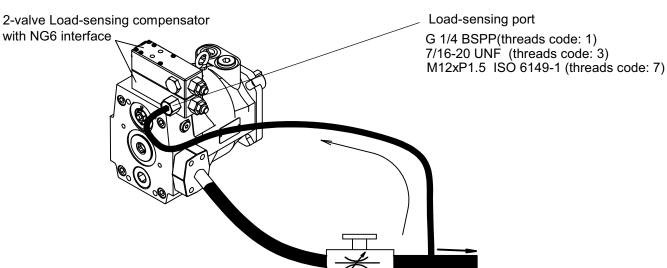


#### VM 2 Valves Load-sensing Compensator with NG6 interface

Two-valve load-sensing compensator with NG6 interface is an option for limiting pressure precisely. By eliminating the impact between pressure and flow, the pump should add two different valves to control flow and pressure.

Version **VM** provides on its top side an interface NG6 which has pressure-adjusted function. If adding a proportional pressure control valve, Electrical Proportional Pressure control is available.







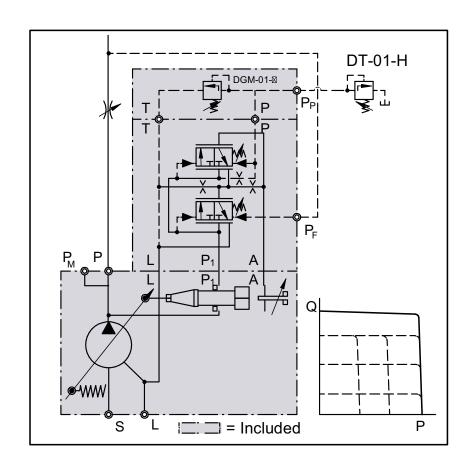
## VA 2 Valves Load-sensing Compensator + Relief Valve

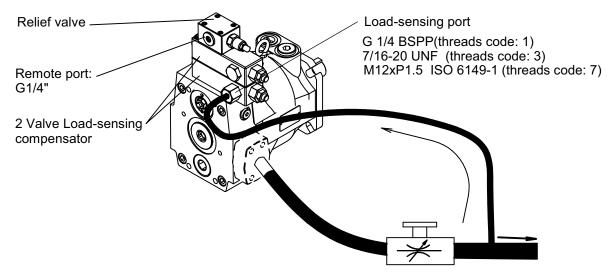
Two-valve load-sensing compensator with NG6 interface is an option for limiting pressure precisely. By eliminating the impact between pressure and flow, the pump should add two different valves to control flow and pressure.

The pilot valve if free to be installed to remote in some distance.

That allows pressure setting e.g. from the control panel of the machine.

The pilot flow supply is internal through the valve spool, and the pilot flow is 1-1.5 L/min.



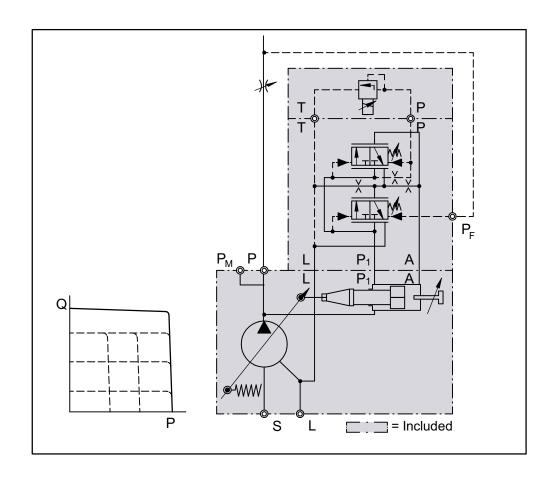


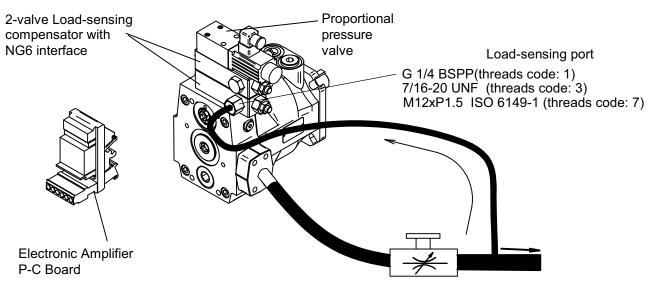


## VJ 2 Valves Load-sensing Compensator + Proportional Pressure Valve

Two-valve load-sensing compensator with NG6 interface is an option for limiting pressure precisely. By eliminating the impact between pressure and flow, the pump should add two different valves to control flow and pressure.

Electrical proportional and pressure control is available by adding a proportional pressure valve on the Load-sensing compensator

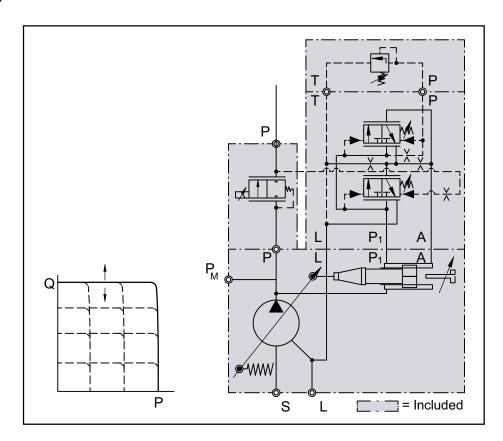


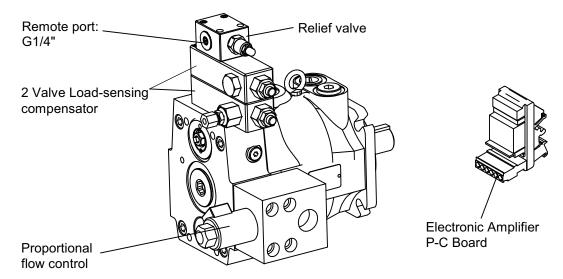




## VQ 2 Valves Load-sensing Compensator + Proportional Flow Valve + Relief Valve

Type **VQ** is an option for limiting pressure precisely. The load-sensing compensator has an external pilot pressure supply. Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant. A variable input speed or a varying load (-pressure) consequently has no influence on the output flow of the pump and the speed of the actuator. Electrical proportional flow control is available by adding a proportional flow control valve.



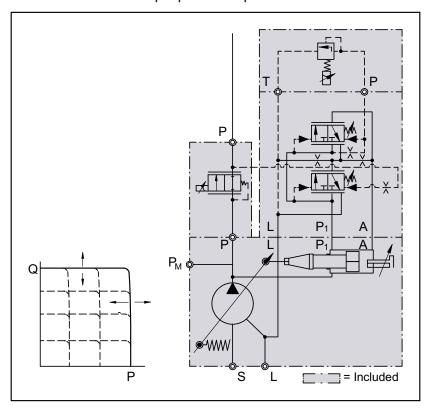


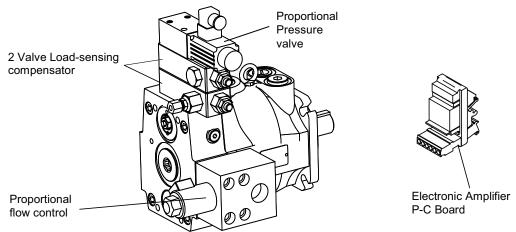


- VK 2 Valves Load-sensing Compensator + Proportional Pressure Valve
  - + Proportional Flow Valve

**VK** control has the same characters as **HK** control for saving energy. It offers the smallest pressure and flow according to the different request.

The displacement is nearly zero when the system stands by, and the motor output is also nearly zero. When the system reaches setting pressure, the pump displacement will reduce by itself. It only needs to add the system required flow, and the pressure remains the same which control the oil temperature. Compared with vane pump, gear pump +PQ valve can save 30-50% energy. The load-sensing compensator+ Proportional flow valve has external pilot pressure supply. Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at the main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow of the pump and the speed of the actuator. Proportional pressure valve is for electrical proportional pressure control.





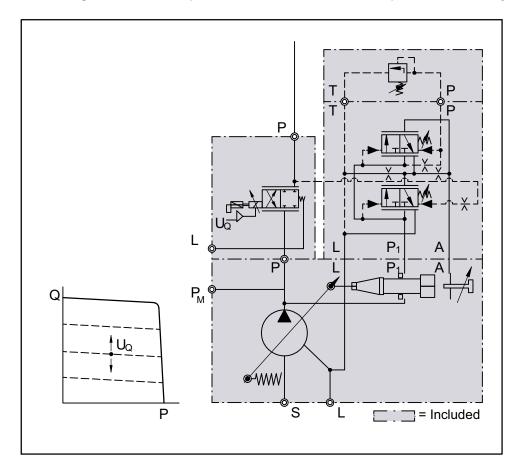


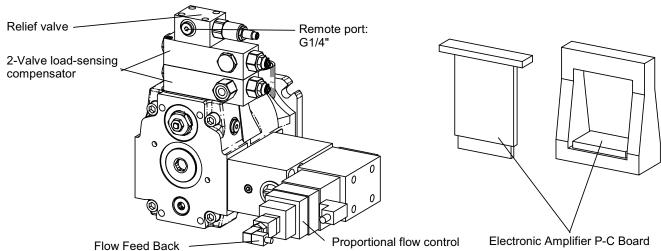
# FV 2 Valves Load-sensing Compensator + High Reacted Proportional Flow Valve + Flow Feed Back + Relief Valve

**FV** control is an option for limiting pressure precisely. By eliminating the impact between pressure and flow, the pump should add two different valves to control flow and pressure.

The electrical control permanently compares input command and actual displacement and powers the proportional flow solenoid of the control valve. A deviation from the commanded displacement leads to a modulation of the input current to the solenoid.

The control valve changes the control pressure until the correct displacement is adjusted.





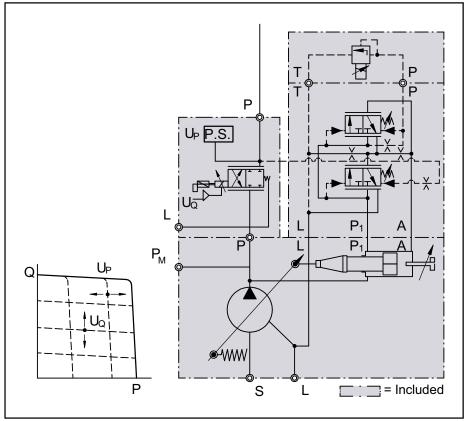


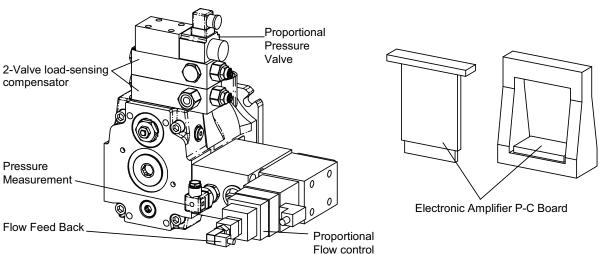
## FG 2 Valves Load-sensing Compensator + High Reacted Proportional Flow Valve

+ Proportional Pressure Valve + Flow & Pressure Feed Back

**FG** control is an option for limiting pressure precisely. By eliminating the impact between pressure and flow, the pump should add two different valves to control flow and pressure. By adding a proportional pressure valve, it would be Electrical Proportional Pressure Control. The electrical control permanently compares input command and actual displacement and powers the proportional flow solenoid of the control valve. A deviation from the commanded displacement leads to a modulation of the input current to the solenoid.

The control valve changes the control pressure until the correct displacement is adjusted. Adding a pressure sensor achieves pressure feedback control.







## PM Horse Power Compensator with NG6 interface

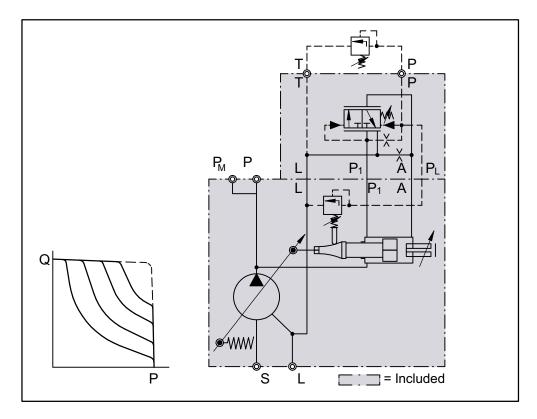
The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve.

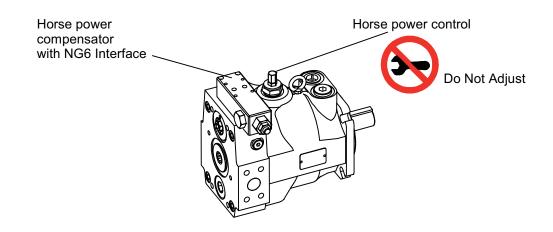
This pilot valve is integrated into the pump and is adjusted by a cam sleeve.

The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting.

At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements. This makes the pump compensate along a constant horse power (torque) curve. Horse power is optional when order. Working pressure can be adjusted by adding pressure leading valve.

Adding the proportional pressure valve achieves the Electrical Proportional Pressure Control.







## PA Horse Power Compensator + Relief Valve

The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve.

This pilot valve is integrated into the pump and is adjusted by a cam sleeve.

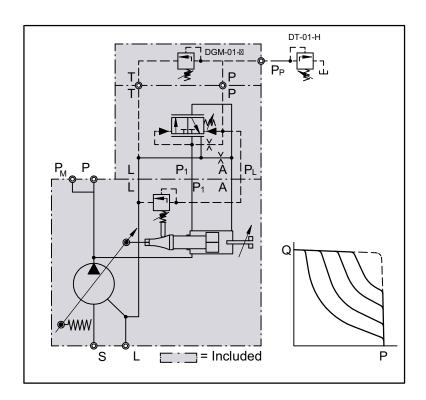
The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting. At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements.

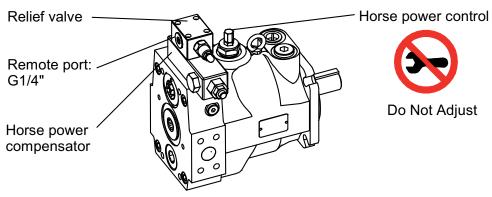
This makes the pump compensate along a constant horse power (torque) curve. Horse power is optional when order.

Working pressure can be adjusted by adding pressure leading valve. Adding the proportional pressure valve achieves the Electrical Proportional Pressure Control.

The pilot valve can be installed remote from the pump in some distance. That allows pressure setting e.g. from the control panel of the machine.

The pilot flow supply is internal through the valve spool and the pilot flow is 1-1.5 L/min.







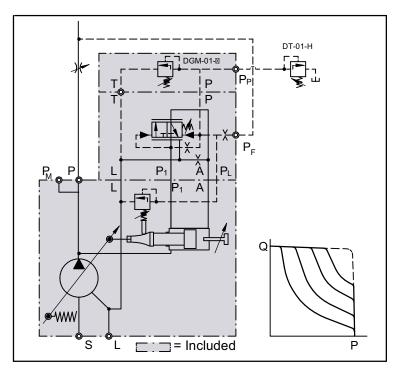
### PH Horse Power Load-sensing Compensator + Relief Valve

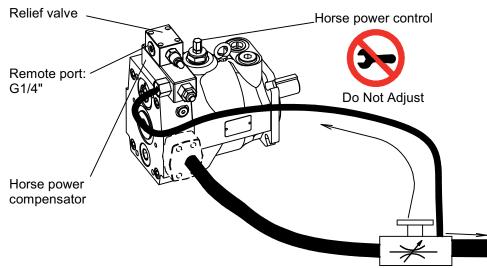
The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve.

This pilot valve is integrated into the pump and is adjusted by a cam sleeve. The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting. At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements. This makes the pump compensate along a constant horse power (torque) curve. Horse power is optional when order.

Working pressure can be adjusted by adding a leading valve on the compensator, and pump flow can also be adjusted on the first pipe by adding an external feedback on the PF port as a control signal on the main stream. The pilot valve can be installed remote from the pump in some distance. That allows pressure setting, e.g. from the contral panel of the machine.

The pilot flow supply is internal through the valve spool, and the pilot flow is 1~1.5 L/min.

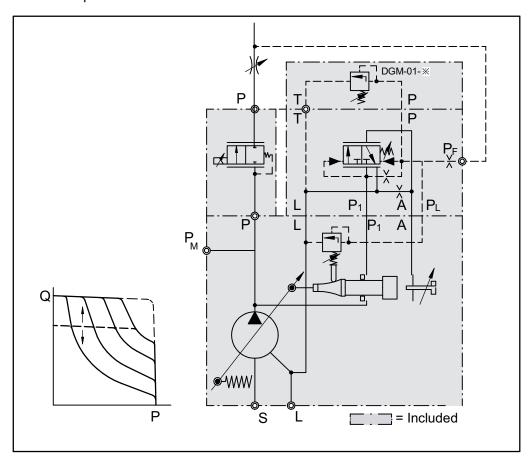


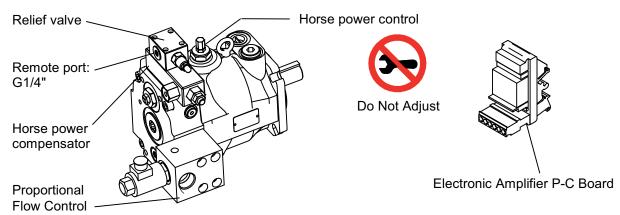




### PQ Horse Power Load-sensing Compensator + Proportional Flow Valve+ Relief valve

The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve. This pilot valve is integrated into the pump and is adjusted by a cam sleeve. The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting. At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements. This makes the pump compensate along a constant horse power (torque) curve. Pressure can be adjusted by adding a leading valve in the compensator, and pump flow can also be adjusted on the first pipe by adding an external feedback on the PF port as a control signal on the main stream. Adding a proportional flow control valve on the P port achieves Electrical Proportional Flow Control.



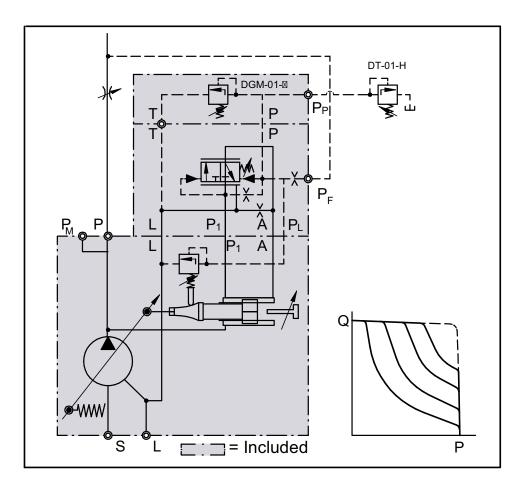


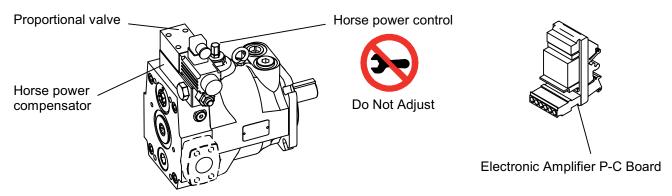


### PJ Horse Power Compensator + Proportional Pressure Valve

The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve. This pilot valve is integrated into the pump and is adjusted by a cam sleeve. The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting. At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements. This makes the pump compensate along a constant horse power (torque) curve.

Pressure-adjusted function is optional by adding a leading proportional pressure valve.



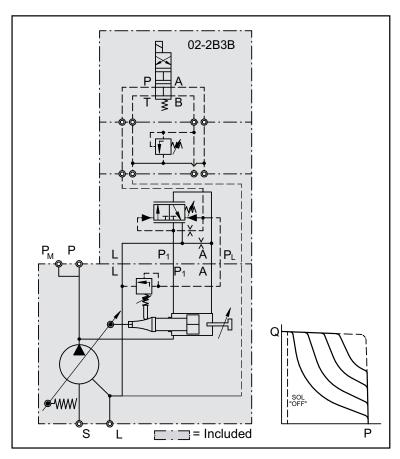


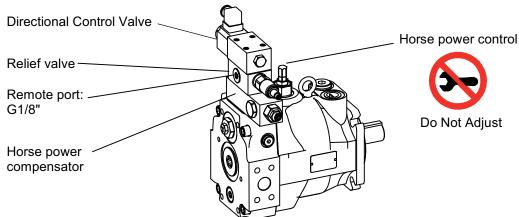


### PR Horse Power Compensator + Electrical Unloading

The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve. This pilot valve is integrated into the pump and is adjusted by a cam sleeve. The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting. At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements.

This makes the pump compensate along a constant horse power (torque) curve. Electrical unloading function is optional by adding an electric directional control valve. This control is suitable for long period of unloading. Oil temperature and noise remain low level through out the electrical unloading function when the system stops working.



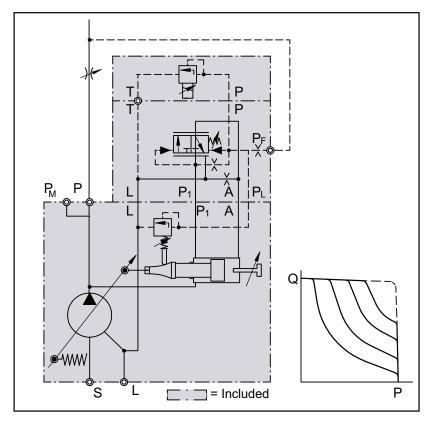


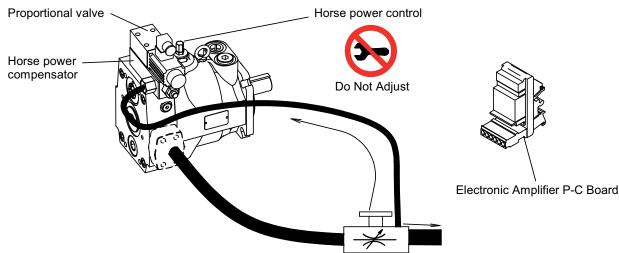


### PS Horse Power Load-sensing Compensator + Proportional Pressure Valve

The hydraulic- mechanical horse power compensator consists of a modified remote pressure compensator or of a modified load-sensing compensator and a pilot valve. This pilot valve is integrated into the pump and is adjusted by a cam sleeve. The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting. At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements. This makes the pump compensate along a constant horse power (torque) curve.

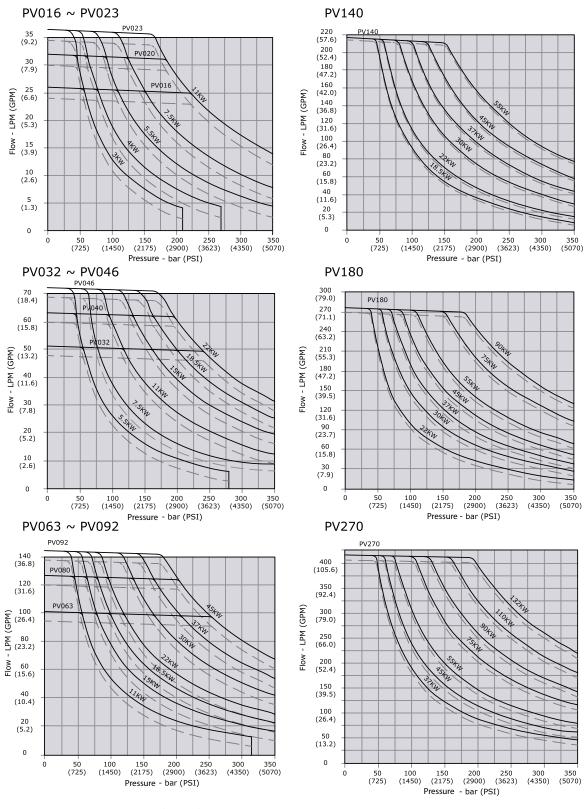
Electrical pressure-adjusted function is optional by adding a leading proportional pressure valve, and pump flow can also be adjusted on the first pipe by adding an external feedback on the PF port as a control signal on the main stream.







### Horse Power Compensator - diagrams



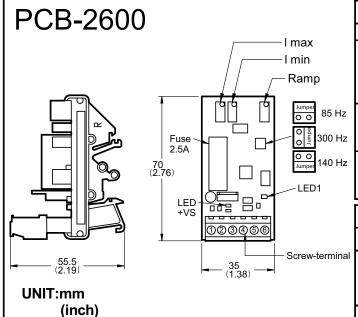
The diagrams are only valid for the following working conditions: speed: n=1500 (- --) and 1800 (-

temperature: t=50°C

fluid: mineral oil HLP, ISO VG46 viscosity: v=46 mm2/s at 40°C



### PCB-2600 Proportional Amplifier



## NOTICE

Do not remove the amplifier from the coil while the power is on.

This will cause a failure in the internal circuits of the amplifier, resulting in loss of output to the coil.

### INSTRUCTIONS FOR SETTING

#### **SUPPLY** Green LED

**RAMP** Ramping up/down time adjustment. For long ramping times, turn potentiometers clockwise, for short ramping times, turn potentiometers counter-clockwise.

### MAX/MIN I max / I min

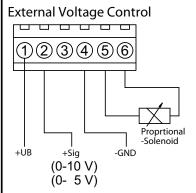
There are multi-course potentiometers for adjustment of min-max and also ramp time.

#### FREQUENCY ADJ.

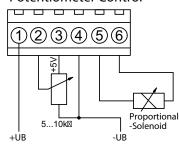
The dither frequency can be set with a Jumper to 85, 140, or 300 Hz.

	TECHIN	ICAL DAT	Ά				
	Supply Voltage:	10-35	VDC				
I	Max. Current : 0-2600 mA adjustable						
	for 12 and 24 VDC						
	(Output is a PWM-DC)						
	Min. Current : 0-600 mA adjustable						
	Ramp Adjustment:	0~5	Sec.				
	Dither Frequency: 85, 14	0, 300 Hz	to				
	be set by jumper(Standard	140 Hz)					
	Ambient Operating		-15~140 °F				
	temperature		-10~60 °C				

### Connections



### Potentiometer Control



- . Clamp Connections plug in connector
- Pin 1 =+ UB; Supply voltage (10-35 VDC)

0.05

Kg

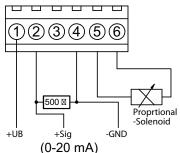
- Pin 2 = control voltage (+ Sig)
- Pin 3 = Auxiliary voltage (+ 5 VDC)
- Pin 4 = Ground (GND)
- Pin 5 = Solenoid (-)
- Pin 6 = Solenoid (+)

#### . Potentiometer

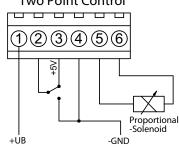
Turn clockwise means increasing current or Extension of ramp time

App. 10 turns for complete range

### External Current Control



### Two Point Control



#### .Fuse

Weight:

Standard 20 mm Glass fuse 2.5 A T

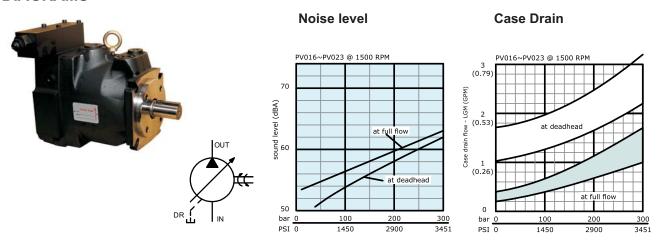
#### .LED's

LED +VS (green) = lights, when voltage supply and fuse are in order

LED1 (red) = lights, if there is an output to the solenoid

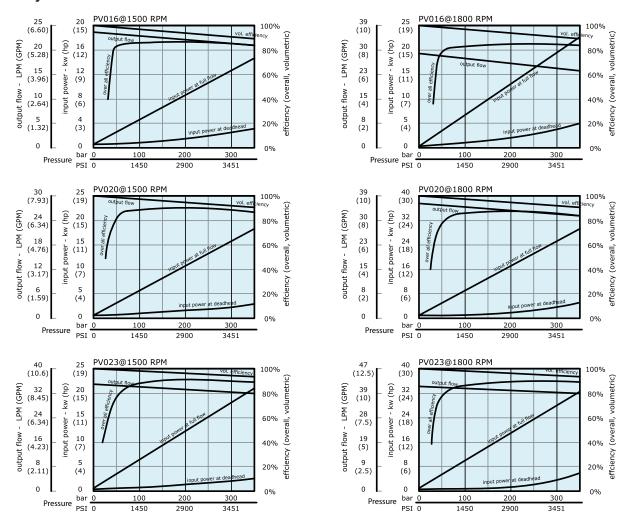


#### **DIAGRAMS**



The efficiency and power graphs are measured at an input speed of n = 1500 RPM, a temperature of  $40^{\circ}$ C and a fluid viscosity of 46 mm2/s. Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min, if at pilot operated compensators (codes  $G^*$ ,  $H^*$ ,  $P^*$ , horse power compensator and p/Qcontrol) the control flow of the pressure pilot valve also goes through the pump. Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 40 l/min. Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

#### **Efficiency**

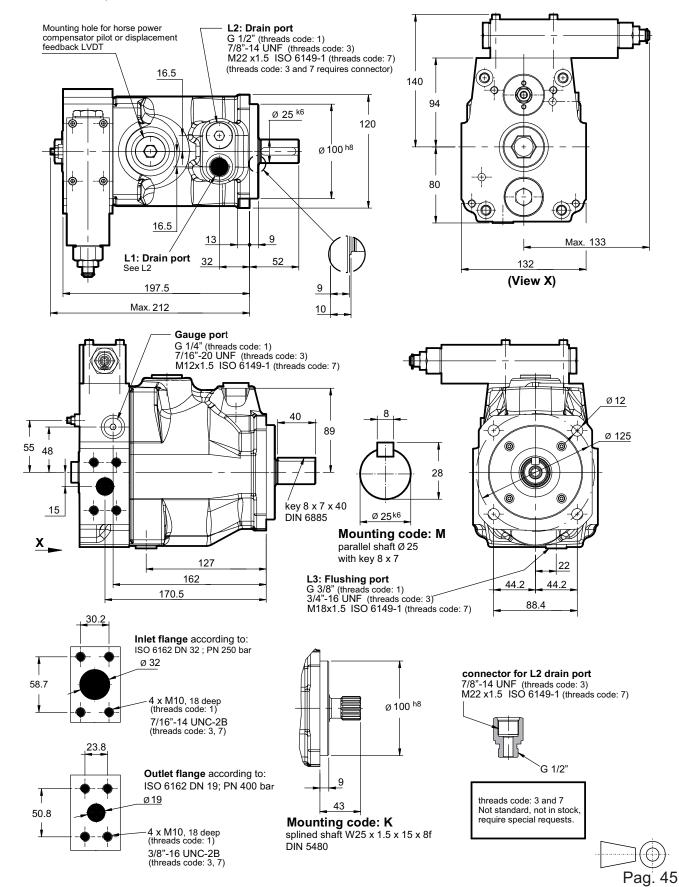


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**Metric version** 

# INSTALLATION DRAWINGS MOUNTING: M, K

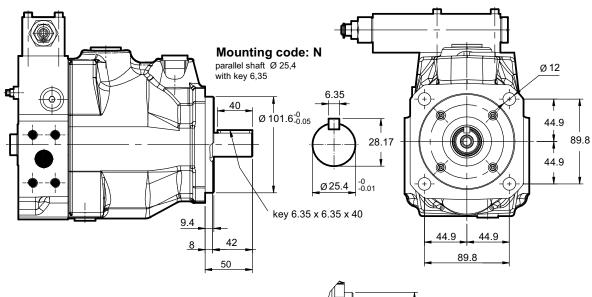




SAE version and thru drive

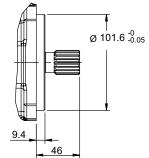
### INSTALLATION DRAWINGS

**MOUNTING: N, D** 

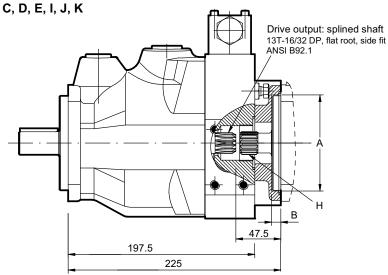


### Mounting code: D

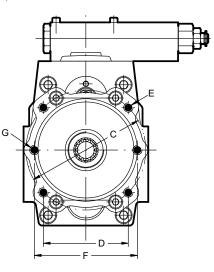
splined shaft 15T 16/32 DP, flat root, side fit ANSI B92.1



### Thru drive:



Thru shaft adaptors are available with the following dimensions:									
Thru code	Α	В	С	D	Е	F	G		
	63	10	85	-	M8	100	M8		
J	80	10	103	-	M8	109	M10		
K	100	10.5	125	-	M10	not avail.	not avail.		
С	50.8	10	-	-	-	82	M8		
D	82.55	10	-	-	-	106	M10		
Е	101.6	10.5	-	89.8	M10	not avail.	not avail.		



Thread codes are 3 and 7, the dimensions E and G are UNC-2B threads

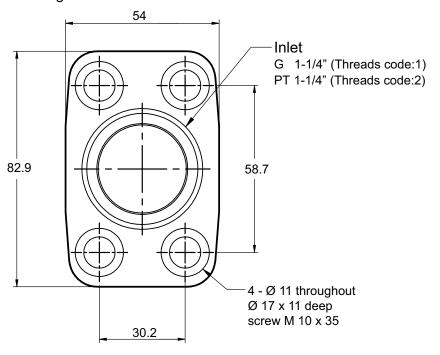
threads code: 3 and 7 Not standard, not in stock, require special requests.

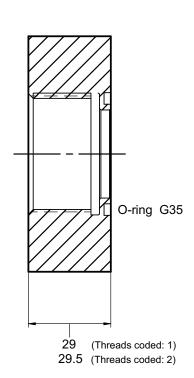




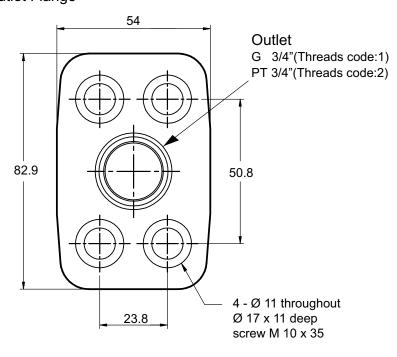
# INSTALLATION DRAWINGS PORT FLANGES

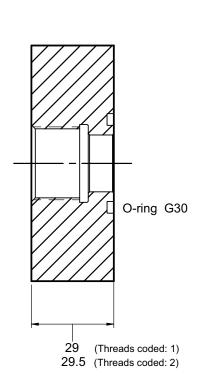
### Inlet Flange





### **Outlet Flange**

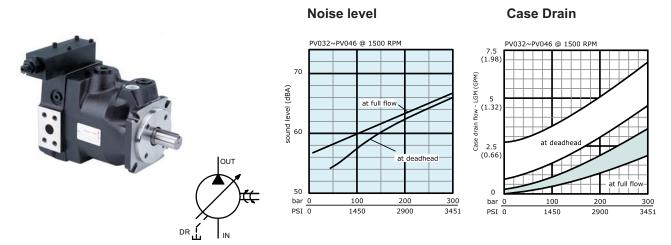






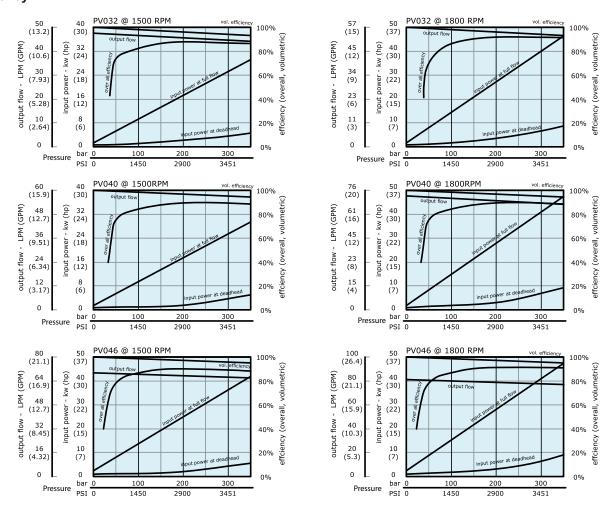


#### **DIAGRAMS**



The efficiency and power graphs are measured at an input speed of n = 1500 RPM, a temperature of  $40^{\circ}$ C and a fluid viscosity of 46 mm2/s. Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min, if at pilot operated compensators (codes  $G^*$ ,  $H^*$ ,  $P^*$ , horse power compensator and p/Qcontrol) the control flow of the pressure pilot valve also goes through the pump. Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 60 l/min. Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

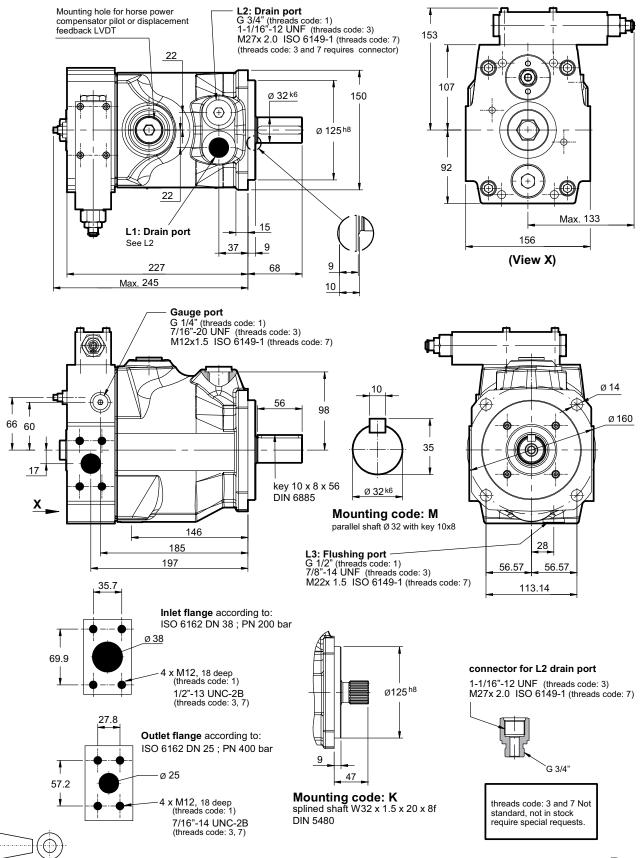
#### **Efficiency**





**Metric version** 

# INSTALLATION DRAWINGS MOUNTING: M, K

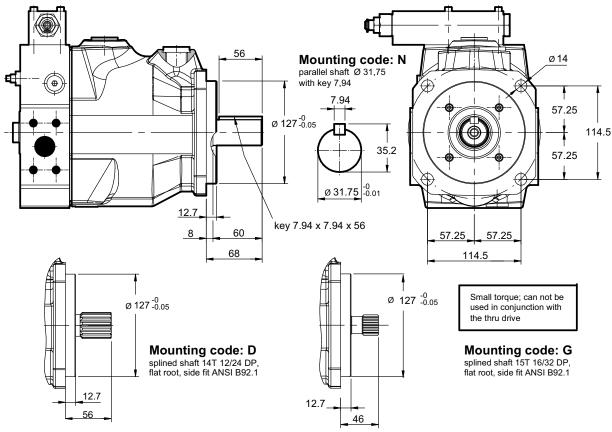


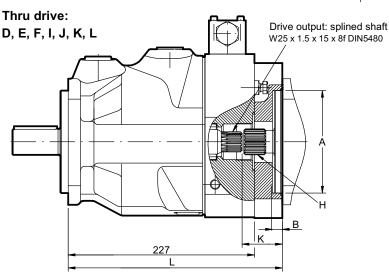


SAE version and thru drive

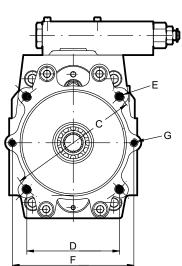
### **INSTALLATION DRAWINGS**

**MOUNTING: N, D, G** 





	Thru shaft adaptors are available with the following dimensions:									
thru code	Α	В	С	D	E	F	G	K	L	
I	63	8.5	85	-	M8	100	M8	49	261	
J	80	8.5	103	-	M8	109	M10	49	261	
К	100	10.5	125	-	M10	140	M12	49	261	
L	125	12	160	-	M12	not avail.	not avail.	49	261	
D	82.55	8	-	-	-	106	M10	49	261	
Е	101.6	11	-	89.8	M10	146	M12	49	261	
F	127	13.5	-	114.5	M12	not avail.	not avail.	64	276	



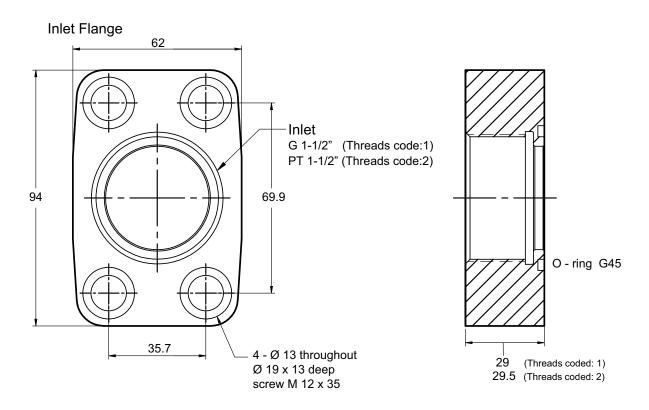
Thread codes are 3 and 7 the dimensions E and G are UNC-2B threads

threads code: 3 and 7 Not standard, not in stock require special requests.





### **INSTALLATION DRAWINGS PORT FLANGES**

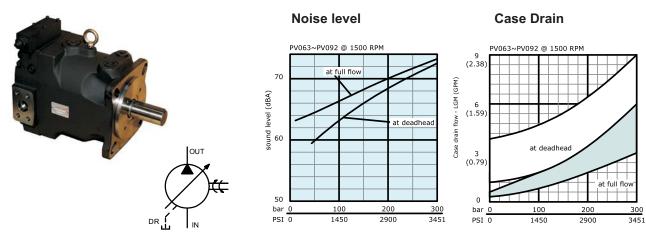


## **Outlet Flange** Outlet G 1" (Threads code:1) PT 1" (Threads code:2) 94 57.2 O-ring G35 4 - Ø 13 throughout Ø 19 x 13 deep screw M 12 x 35 27.8 29 (Threads coded: 1) 29.5 (Threads coded: 2)



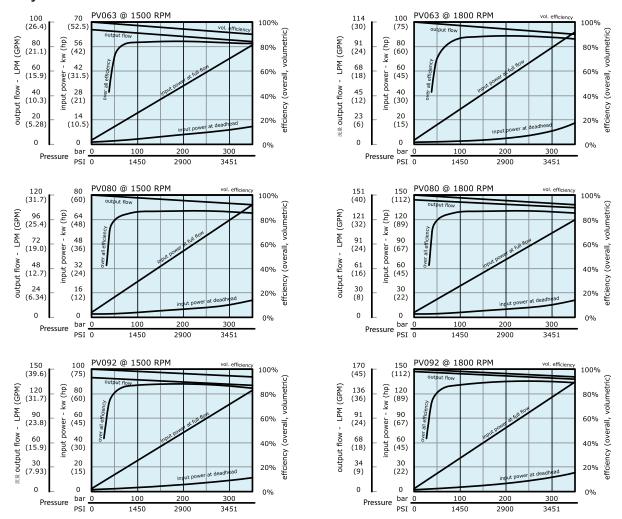


#### **DIAGRAMS**



The efficiency and power graphs are measured at an input speed of  $n=1500\ RPM$ , a temperature of  $40^{\circ}C$  and a fluid viscosity of  $46\ mm2/s$ . Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to  $1.2\ l/min$ , if at pilot operated compensators (codes  $G^*$ ,  $H^*$ ,  $P^*$ , horse power compensator and p/Q+control) the control flow of the pressure pilot valve also goes through the pump. Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to  $80\ l/min$ . Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

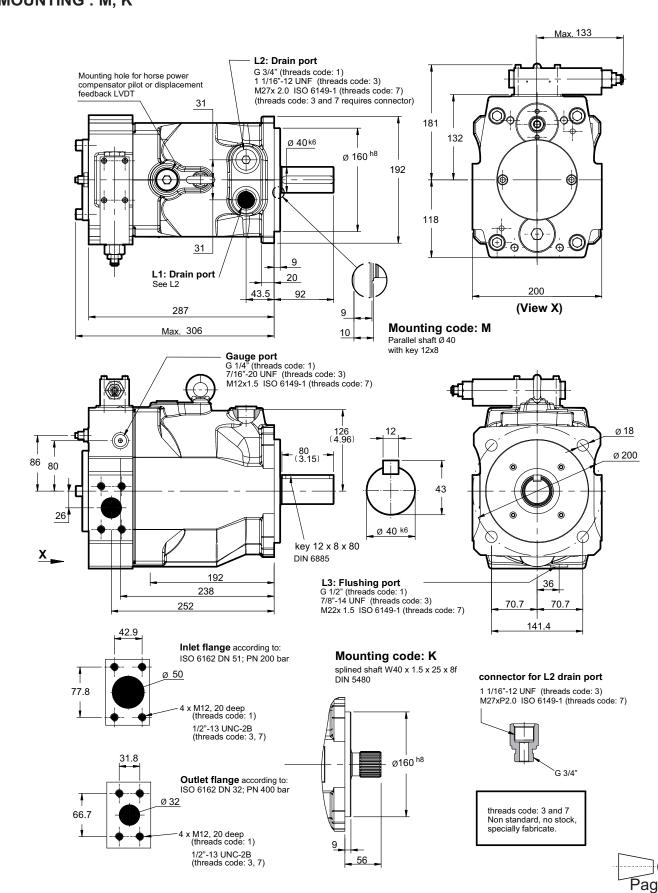
#### **Efficiency**





**Metric version** 

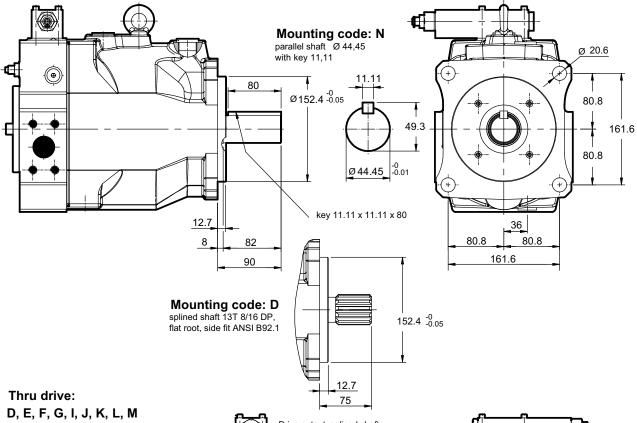
INSTALLATION DRAWINGS MOUNTING: M, K

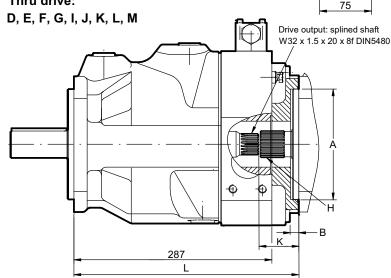




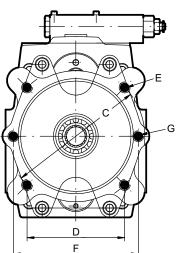
SAE version and thru drive

# INSTALLATION DRAWINGS MOUNTING: N, D





Thru shaft adaptors are available with the following dimensions:									
Thru code	Α	В	С	D	E	F	G	K	L
- 1	63	10	85	-	M8	100	M8	58	326
J	80	10	103	-	M8	109	M10	58	326
K	100	12	125	-	M10	140	M12	58	326
L	125	12	160	-	M12	180	M16	58	326
М	160	12	200	-	M16	not avail.	not avail.	58	326
D	82.55	10	-	-	-	106	M10	58	326
E	101.6	12	-	89.8	M10	146	M12	58	326
F	127	14	-	114.5	M12	181	M16	58	326
G	152.4	14	-	161.6	M16	not avail.	not avail.	78	346



Thread codes are 3 and 7 the dimensions E and G are UNC-2B threads

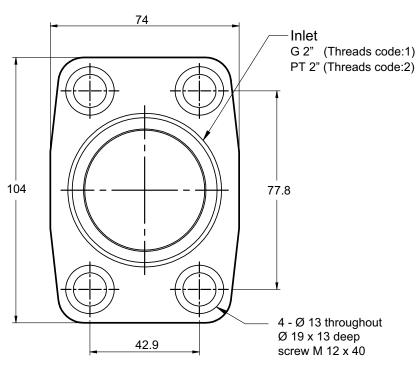
threads code: 3 and 7 Not standard, not in stock require special requests.

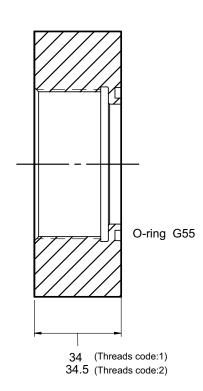




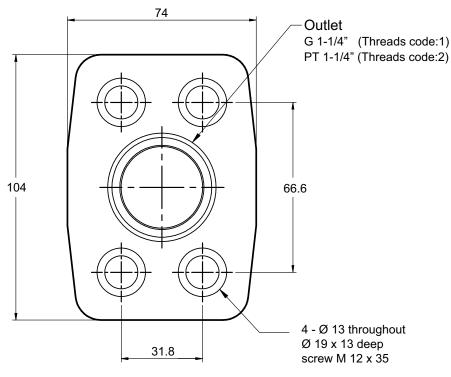
### **INSTALLATION DRAWINGS PORT FLANGES**

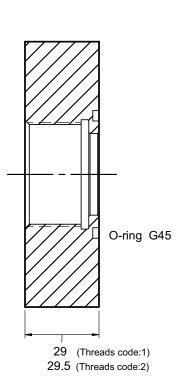
### Inlet Flange





### **Outlet Flange**

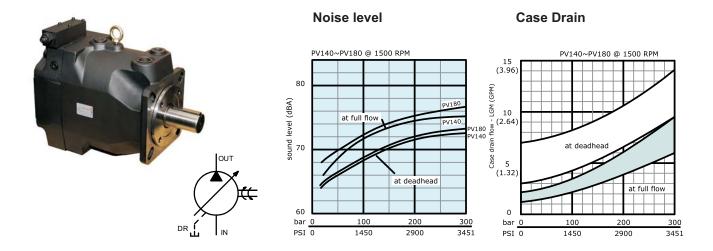








# **Size 4 - PV 140, PV 180** DIAGRAMS



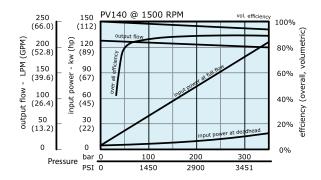
The efficiency and power graphs are measured at an input speed of n = 1500 RPM, a temperature of  $40^{\circ}$ C and a fluid viscosity of 46 mm2/s.

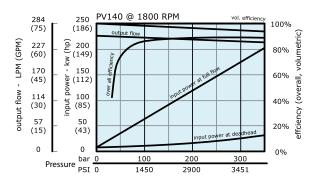
Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min, if at pilot operated compensators (codes G\*, H\*, P\*, horse power compensator and p/Q,control) the control flow of the pressure pilot valve also goes through the pump.

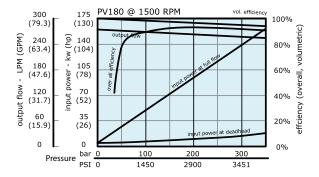
Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port.

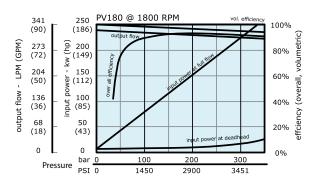
This dynamic control flow can reach up to 40 l/min. Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

#### **Efficiency**







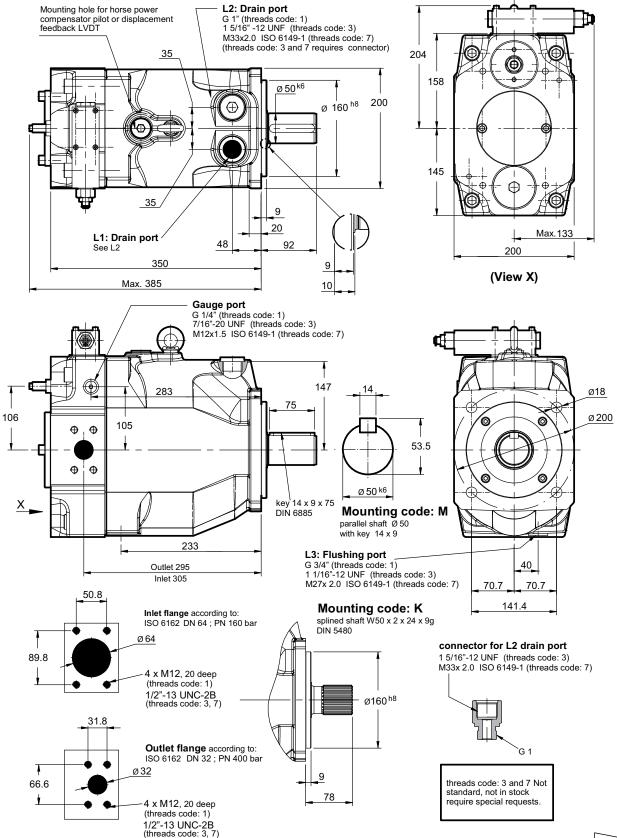




### Size 4 - PV 140, PV 180

**Metric version** 

## INSTALLATION DRAWINGS MOUNTING: M, K

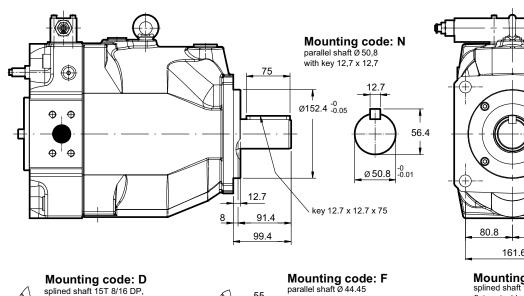


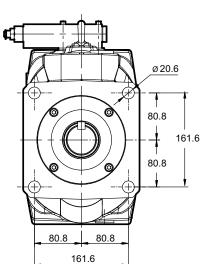


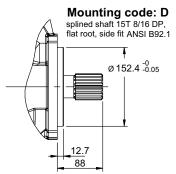
### Size 4 - PV 140, PV 180

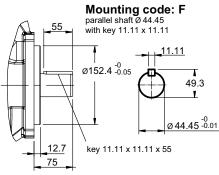
SAE version and thru drive

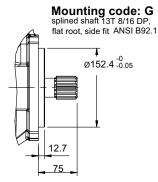
# INSTALLATION DRAWINGS MOUNTING: N, F, D, G

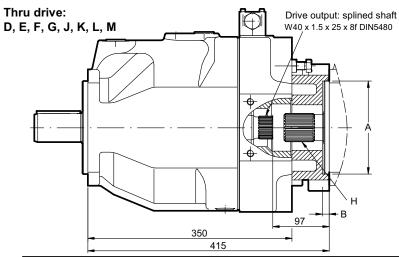




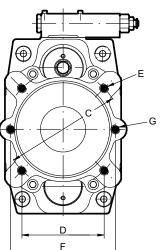








410									
	Thru shaft adaptors are available with the following dimensions:								
Thru code	Α	В	С	D	Е	F	G		
J	80	10	103	-	M8	109	M10		
K	100	12	125	-	M10	140	M12		
L	125	12	160	-	M12	180	M16		
М	160	12	200	-	M16	not avail.	not avail.		
D	82.55	10	-	-	-	106	M10		
Е	101.6	12	-	89.8	M10	146	M12		
F	127	14	-	114.5	M12	181	M16		
G	152.4	14	-	161.6	M16	not avail.			



Thread codes are 3 and 7 the dimensions E and G are UNC-2B threads

threads code: 3 and 7 Not standard, not in stock require special requests.

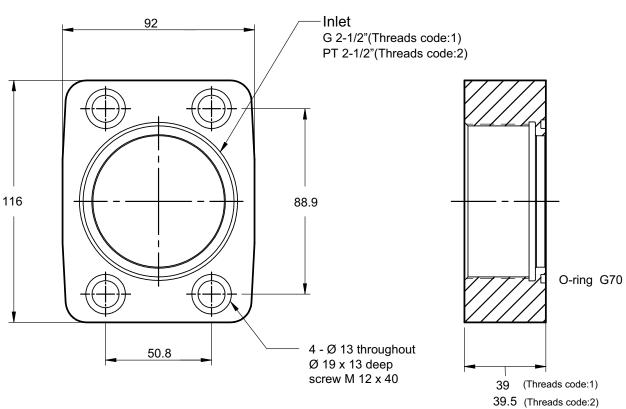




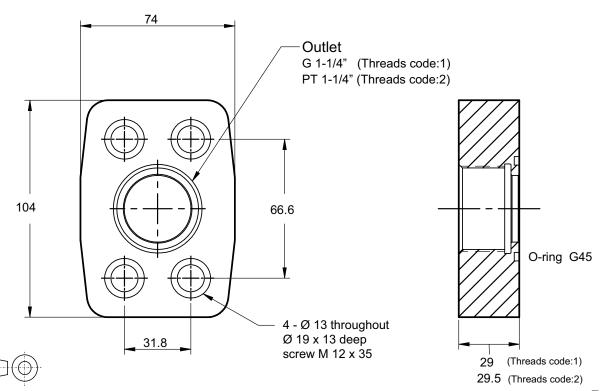
### Size 4 - PV 140, PV 180

# INSTALLATION DRAWINGS PORT FLANGES

### Inlet Flange



### **Outlet Flange**

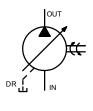




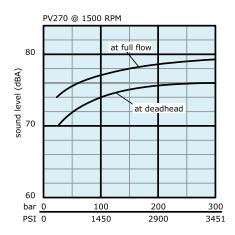
### Size 5 - PV 270

### **DIAGRAMS**

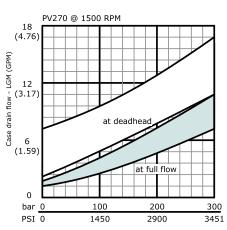




#### Noise level

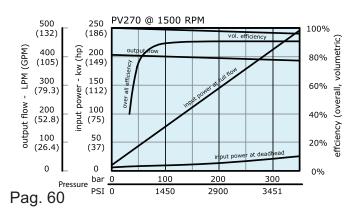


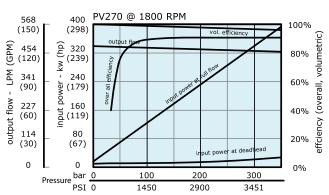
#### **Case Drain**



The efficiency and power graphs are measured at an input speed of n = 1500 RPM, at temperature of  $40^{\circ}$ C and a fluid viscosity of  $46 \text{ mm}^2$ /s. Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min , if at pilot operated compensators (codes  $G^*$ ,  $H^*$ ,  $P^*$ , horse power compensator and p/Q+control) the control flow of the pressure pilot valve also goes through the pump. Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 120 l/min. Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

### **Efficiency**



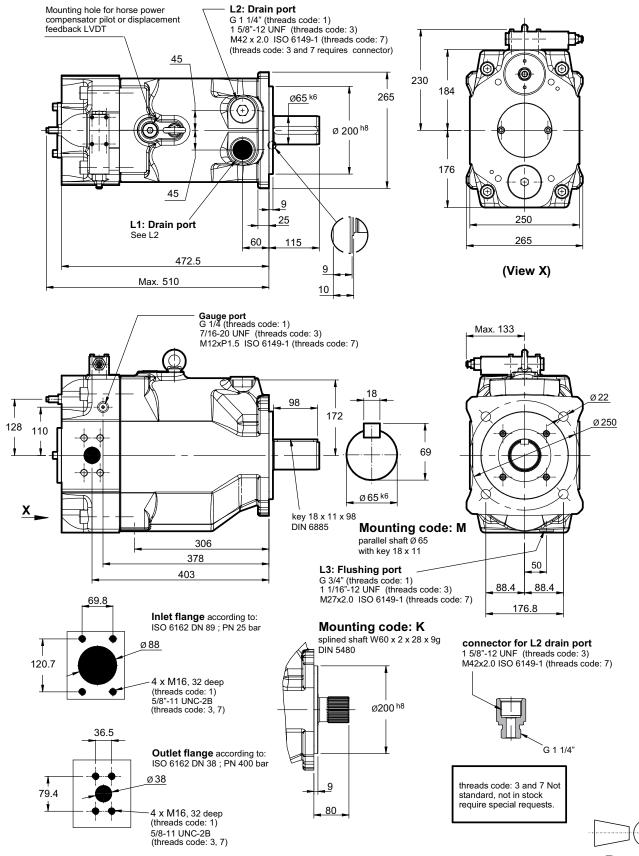




### Size 5 - PV 270

**Metric version** 

## INSTALLATION DRAWINGS MOUNTING: M, K

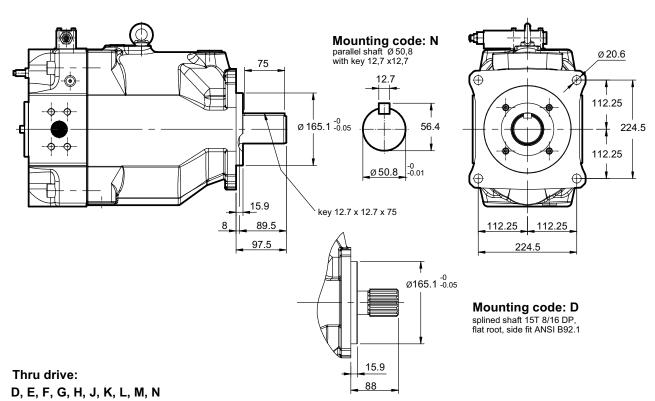


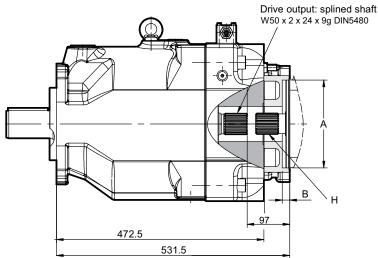


### Size 5 - PV 270

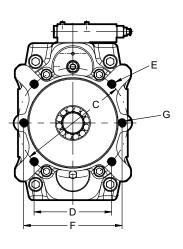
SAE version and thru drive

# INSTALLATION DRAWINGS MOUNTING: N, D





					_				
Thru shaft adaptors are available with the following dimensions:									
Thru code	Α	В	С	D	Е	F	G		
J	80	8.5	103	-	M8	109	M10		
K	100	10.5	125	-	M10	140	M12		
L	125	10.5	160	-	M12	180	M16		
М	160	13.5	200	-	M16	224	M20		
N	200	13.5	250	-	M20	not avail.	not avail.		
D	82.55	8	-	-	-	106	M10		
E	101.6	11	-	89.8	M10	146	M12		
F	127	13.5	-	114.5	M12	181	M16		
G	152.4	13.5	-	161.6	M16	229	M20		
Н	165.1	17	-	224.5	M20	not avail.	not avail.		



Thread codes are 3 and 7 the dimensions E and G are UNC-2B threads

threads code: 3 and 7 Not standard, not in stock require special requests.



# Variable Displacement Open Loop Circuit Axial Piston Pump

**PV Series** 

#### GENERAL INSTALLATION INFORMATION

#### Fluid recommendations

Premium quality hydraulic mineral oil fluids are recommended, like H-LP oils to DIN 51524, part2. The viscosity range should be 25 to 50 mm<sup>2</sup>/s (cSt) at 50° C. Operating temp. –10 to +70°C. For other fluids such as phosphoric acid esters or for other operating conditions, please consult our Tech. Dept. for assistance.

#### Seals

NBR (Nitrile) seals are used for operation with hydraulic fluids based on mineral oil. For synthetic fluid, as perhaps phosphoric acid esters, Fluorocarbon seals are required. Please consult our Tech. Dept. for assistance.

### **Filtration**

For maximum pump and system component functionability and life, the system should be protected from contamination by effective filtration.

Fluid cleanliness should be in accordance with ISO classification ISO 4406.

The quality of filter elements should be in accordance with ISO standards.

- (1) Minimum requirement for filtration rate ×(mm): General hydraulic systems for satisfactory operation:
  - Class 19/15, to ISO 4406 X=25 $\mu$ m ( $\beta$ 25  $\geq$  75) to ISO 4572
- (2) Hydraulic systems with maximum component life and functionability: Class 16/13, to ISO 4406 X=10 $\mu$ m ( $\beta$ 10  $\geq$  75) to ISO 4572

It is recommended to use return line or pressure filters.

We can offers a wide range of these filters for all common applications and mounting styles.

The use of suction filters should be avoided, especially with fast response pumps.

Bypass filtration is a good choice for best filter efficiency.

### Installation and mounting

Horizontal mounting:

Outlet port-side or top. Inlet port-side or bottom, drain port always uppermost.

Vertical mounting: Shaft pointing upwards.

Install pump and suction line in such way that the maximum inlet vacuum never exceeds 0.8 bar absolute. The inlet line should be as short and as straight as possible. A short suction line cut to 45° is recommended when the pump is mounted inside the reservoir, to improve the inlet conditions. All connections should be leak-free, otherwise the air in the suction line will cause cavitations, noise, and damage to the pump.

### Shaft rotation and alignment

Pump and motor shafts must be aligned within 0.25mm T.I.R. maximum.

A floating coupling must be used.

Bell-housings and couplings can be ordered following the specific catalog.

Please follow the coupling manufacturer's installation instructions.

Please consult our Tech. Dept for assistance. on radial load type drives.

### Start up

Prior to start up, the pump case must be filled with hydraulic fluid (use the upper case drain port). Initial start up should be at zero pressure with an open circuit to enable the pump to prime. Pressure should only be increased once the pump has been fully primed.

Attention: Check motor rotation direction.



# Variable Displacement Open Loop Circuit Axial Piston Pump

### **GENERAL INSTALLATION INFORMATION** (continued)

### Operating noise of pumps

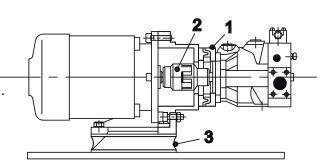
The normal operating noise of a pump and constantly-operation noise of the entire hydraulic system is largely determined by where and how the pump is mounted and how it is connected to the down stream hydraulic system. Besides, size, style, and installation of hydraulic tube are the major influence on the overall noise emitted by a hydraulic system.

#### Noise reduction measures

Flexible elements help to prevent pump body vibration from being transmitted to other construction elements, where amplification may occur. Such elements can be:

Bell housing with elastic dampening flange with vulcanized labyrinth

- (1) Floating and flexible coupling
- (2) Damping rails
- (3) Or silent blocks for mounting the electric motor or the foot mounting flange
- (4) Flexible tube connections (compensators) or hoses on inlet, outlet, and drain port of the pump.
- (5) Exclusive use of gas tight tube fittings for inlet connections to avoid ingression of air causing cavitations and excessive noise.



#### **Drain line**

The drain line must lead directly to the reservoir without restriction. The drain line must not be connected to any other return line. The end of the drain line must be below the lowest fluid level in the reservoir and as far away as possible from the pump inlet line. This ensures that the pump is not empty itself when it's not in operation and the hot aireated oil will not be recirculated. For the same reason, when the pump is mounted inside the reservoir, the drain line should be arranged in such a way that a siphon is created. This ensures that the pump is always filled with fluid. The drain pressure must not exceed 1 bar. Drain line length should not exceed 2 meters. Minimum diameter should be selected according to the port size and a straight low pressure fitting with maximized bore should be used.

	PV016~PV023	PV032~PV046	PV063~PV092	PV140~PV180	PV270
Size of pipe joints	3/8(φ8.5 or more)	1/2(φ12 or more)	3/4(φ16 or more)	1(φ19 or more)	1-1/4(φ22 or more)
I.D. of pipes	φ12 more	φ15 more	φ19 more	φ25 more	φ32 more
Length of drain	Under 1m	Under 1m	Under 1m	Under 1m	Under 1m



### **GENERAL INSTALLATION INFORMATION** (continued)

### Thru Drive, Shaft load limitation

The max. Transferable torque in Nm for the different shafts options are:

Shaft code	PV16-23	PV32-46	PV63-125	PV140-180	PV270
Shall code	PV10-23	PV56&65	P V 03-125	PV 140-160	PV2/U
N	300	550	1320	2000	2000
D	300	610	1218	2680	2680
F	-	-	-	1320	-
G	-	-	-	1640	-
M	300	570	1150	1900	2850
K	405	675	1400	2650	3980

### Important notice

The max. allowable torque of the individual shaft must not be exceeded.

For 2-pump combinations, there is no problem because PV series offers 100% thru torque. For 3-pump combinations more), the limit torque will be reached or exceeded.

Therefor, it is necessary to calculate the torque factor and compare with the allowed torque limit factor in the table.

To make the necessary calculations easier and more user friendly it is not required to calculate actual torque requirements in Nm and compare them with the shaft limitations. The table on the right shows limit factors that include material specification, safety factors and conversion factors.

pump	shaft	torque limit factor
	N	17700
PV016-PV023	D	17700
P V U 10-P V U 23	М	17700
	K	20130
	Ν	32680
PV32-46	D	36380
PV56&65	М	33810
	K	40250
	Ν	77280
PV063-PV092	D	72450
P V U 0 3 - P V U 9 2	М	67620
	K	83720
	Ν	118400
	D	158760
PV140-PV180	F	78750
FV 140-FV 160	G	97650
	М	113400
	K	157500
	Ν	119000
PV270	D	159700
FVZIU	М	170100
	K	236250

The total torque factor is represented by the sum of the individual torque factors of all pumps in the complete pump combination.

The torque factor of each individual pump is calculated by multiplying the max.operating pressure p of the pump (bar)with the max.displacement Vg of the pump(cm³/rev).